

FORM PTO-1390
(REV 3/2001)

U.S. DEPARTMENT OF COMMERCE PATENT AND TRADEMARK OFFICE

**TRANSMITTAL LETTER TO THE UNITED STATES
DESIGNATED/ELECTED OFFICE (DO/EO/US)
CONCERNING A FILING UNDER 35 U.S.C. 371**

DATE: March 29, 2002

EXPRESS MAIL LABEL NO.
EL 914619133 USATTORNEY DOCKET NO.
39373/HAC/G602

U.S. APPLICATION NO.

To Be Assigned
10/089577INTERNATIONAL APPLICATION NO.
PCT/US00/27043INTERNATIONAL FILING DATE
29 September 2000PRIORITY DATE CLAIMED
29 September 1999

TITLE OF INVENTION

HORIZONTAL DRILL PIPE RACKER AND DELIVERY SYSTEM

APPLICANT(S) FOR DO/EO/US

Bolding, Vance E.; Brooks, III Andrew J.; and Keen, Michael D.

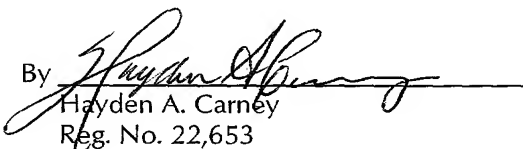
Applicant herewith submits to the United States Designated/Elected Office (DO/EO/US) the following items and other information:

1. ☒ This is a **FIRST** submission of items concerning a filing under 35 U.S.C. 371.
2. ☐ This is a **SECOND** or **SUBSEQUENT** submission of items concerning a filing under 35 U.S.C. 371.
3. ☒ This is an express request to begin national examination procedures (35 U.S.C. 371(f) at any time rather than delay examination until the expiration of the applicable time limit set in 35 U.S.C. 371(b) and PCT Articles 22 and 39(1).
4. ☒ A proper Demand for International Preliminary Examination was made by the 19th month from the earliest claimed priority date.
5. ☒ A copy of the International Application as filed (35 U.S.C. 371(c)(2)).
 - a. ☐ is transmitted herewith (required only if not transmitted by the International Bureau).
 - b. ☐ has been transmitted by the International Bureau.
 - c. ☒ is not required, as the application was filed in the United States Receiving Office (RO/LUS).
6. ☐ A translation of the International Application into English (35 U.S.C. 371(c)(2)).
7. ☒ A copy of the International Search Report (PCT/ISA/210).
8. ☒ Amendments to the claims of the International Application under PCT Article 19 (35 U.S.C. 371(c)(3)).
 - a. ☐ are transmitted herewith (required only if not transmitted by the International Bureau).
 - b. ☐ have been transmitted by the International Bureau.
 - c. ☐ have not been made; however, the time limit for making such amendments has NOT expired.
 - d. ☒ have not been made and will not be made.
9. ☐ A translation of the amendments to the claims under PCT Article 19 (35 U.S.C. 371(c)(3)).
10. ☒ An oath or declaration of the inventor(s) (35 U.S.C. 371(c)(4)). (V. E. Bolding only; Keen and Brooks declarations to follow)
11. ☒ A copy of the International Preliminary Examination Report (PCT/IPEA/409).
12. ☐ A translation of the annexes to the International Preliminary Examination Report under PCT Article 36 (35 U.S.C. 371(c)(5)).

Items 13 to 20 below concern document(s) or other information included:

13. ☒ An Information Disclosure Statement under 37 CFR 1.97 and 1.98.
14. ☒ An assignment document for recording. A separate cover sheet in compliance with 37 CFR 3.28 and 3.31 is included.
15. ☒ A **FIRST** preliminary amendment.
16. ☐ A **SECOND** or **SUBSEQUENT** preliminary amendment.
17. ☐ A substitute specification.
18. ☐ A change of power of attorney and/or address letter.
19. ☐ SMALL ENTITY Assertion: Applicant(s) and any other associated with it/them under 37 CFR § 1.27(a) are a small entity.
20. ☒ Certificate of Mailing by Express Mail.
21. ☒ Other items or information: Letter Under MPEP 1893.03(g)

10089577 032202

U.S. APPLICATION NO. (If known, see 37 CFR 1.5) 10/089577		INTERNATIONAL APPLICATION NO. PCT/US00/27043		ATTORNEY DOCKET NO. 39373/HAC	
21. The following fees are submitted:				CALCULATIONS	PTO USE ONLY
<input type="checkbox"/> Neither international preliminary examination fee (37 CFR 1.482) nor international search fee (37 CFR 1.445(a)(2)) paid to USPTO and International Search Report not prepared by the EPO or JPO: \$1,040.00					
<input type="checkbox"/> International preliminary examination fee (37 CFR 1.482) not paid to USPTO but International Search Report prepared by the EPO or JPO: \$890.00					
<input type="checkbox"/> International preliminary examination fee (37 CFR 1.482) not paid to USPTO but international search fee (37 CFR 1.445(a)(2)) paid to USPTO: \$740.00					
<input type="checkbox"/> International preliminary examination fee paid to USPTO (37 CFR 1.482) but all claims did not satisfy provisions of PCT Article 33(1)-(4): \$710.00					
<input checked="" type="checkbox"/> International preliminary examination fee paid to USPTO (37 CFR 1.482) and all claims satisfied provisions of PCT Article 33(1)-(4): \$100.00				\$100	
ENTER APPROPRIATE BASIC FEE AMOUNT =				\$100	
Surcharge of \$130 for furnishing the oath or declaration later than <input type="checkbox"/> 20 <input type="checkbox"/> 30 months from the earliest claimed priority date (37 CFR 1.492(e)).				\$	
Claims	Number Filed	Number Extra	Rate		
Total Claims	53 -20=	33	X \$18	\$594	
Independent Claims	8 -3=	5	X \$84	\$420	
Multiple dependent claim(s) (if applicable)			+ \$280	\$	
TOTAL OF ABOVE CALCULATIONS =				\$1114	
Reduction by 1/2 for filing by small entity, if applicable. Verified Small entity statement must also be filed. (Note 37 CFR 1.9, 1.27, 1.28).				\$	
SUBTOTAL =				\$1114	
Processing fee of \$130 for furnishing the English translation later than <input type="checkbox"/> 20 <input type="checkbox"/> 30 months from the earliest claimed priority date (37 CFR 1.492(f)).				\$	
TOTAL NATIONAL FEE =				\$1114	
Fee for recording the enclosed assignment (37 CFR 1.21(h)). The assignment must be accompanied by an appropriate cover sheet (37 CFR 3.28, 3.31). \$40.00 per property				\$ 40	
TOTAL FEES ENCLOSED =				\$1154	
Note (1): The basic national fee must be paid when filing this application. The 20-month time limit (37 CFR § 1.494) and 30-month time limit (37 CFR § 1.495) are not extendable.				Amount to be: refunded	\$
				charged	\$
<p>a. <input checked="" type="checkbox"/> A check in the amount of \$1154 to cover the above fees is enclosed.</p> <p>b. <input type="checkbox"/> Please charge my Deposit Account No. _____ in the amount of \$ _____ to cover the above fees. A duplicate copy of this sheet is enclosed.</p> <p>c. <input checked="" type="checkbox"/> The Commissioner is hereby authorized to charge any additional fees which may be required, or credit any overpayment to Deposit Account No. <u>03-1728</u>. A duplicate copy of this sheet is enclosed.</p> <p>NOTE (2): Where an appropriate time limit under 37 CFR 1.494 or 1.495 has not been met, a petition to revive (37 CFR 1.137(a) or (b)) must be filed and granted to restore the application to pending status.</p> <p>SEND ALL CORRESPONDENCE TO:</p> <p>Hayden A. Carney CHRISTIE, PARKER & HALE P.O. Box 7068 Pasadena, CA 91109-7068</p> <p>CUSTOMER NUMBER: 23363</p> <p>By  Hayden A. Carney Reg. No. 22,653</p>					

PATENT

IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

EXPRESS MAIL NO. EL 914619133 US

Applicant : Bolding, Vance E. et al.
 Application No. : Unassigned (National Stage of PCT/US00/27043)
 Filed :
 Title : HORIZONTAL DRILL PIPE RACKER AND
 DELIVERY SYSTEM
 Grp./Div. : 3652
 Examiner : Janice L. Krizek
 Docket No. : 39373/HAC/G602

LETTER UNDER MPEP 1893.03(g)

Assistant Commissioner for Patents
 Washington, D.C. 20231

Post Office Box 7068
 Pasadena, CA 91109-7068
 March 29, 2002

Commissioner:

This letter pertains to the application described above which is filed concurrently herewith under 35 U.S.C. 371 as the US national stage of PCT/US00/27043. The documents comprising that application include a copy of the International Search Report (ISR) issued by ISA/US. The application documents also include a copy of the International Preliminary Examination Report (IPER) issued by IPEA/US in the course of examination of the international stage application under PCT Chapter II. The IPER found all claims presented in the national stage application to meet the patentability criteria of PCT Article 33(2)-(4).

In view of the content of MPEP 1893.03(g), applicant understands that in its consideration of this national stage application the Office will consider each of the references identified in the ISR, and that applicants need not (and preferably, in consideration of the size of the Office's file of the national stage application, should not) file copies of those references to the office. However, applicants can and will promptly provide copies of those references on request by the Office.

Applicant also notes 37 CFR § 1.98(c), the provision of which are met because the international stage application complies with subparagraph (1) thereof and the enclosed ISR is the full equivalent of an IDS complying with subparagraph (2) thereof.

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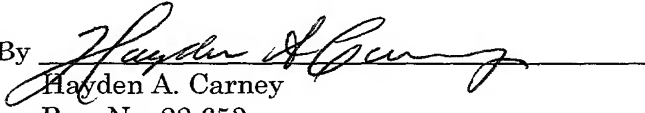
JC13 Rec'd PCT/PTC 29 MAR 2002

Application No. Unassigned (National Stage of PCT/US00/27043)

An IDS is filed herewith regarding references cited in the specification of this application but not cited in the ISR.

Respectfully submitted,

CHRISTIE, PARKER & HALE, LLP

By 
Hayden A. Carney
Reg. No. 22,653
626/795-9900

HAC/sad

Enclosures:

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10089577-032502

PATENT

IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

EXPRESS MAIL NO. EL 914619133 US

Applicant : Vance E. Bolding et al.
Application No. : Not Assigned
Filed : March 29, 2002
Title : HORIZONTAL DRILL PIPE RACKER AND
DELIVERY SYSTEM
Docket No. : 39373/HAC/G602

PRELIMINARY AMENDMENT

Assistant Commissioner for Patents
Washington, D.C. 20231

Post Office Box 7068
Pasadena, CA 91109-7068
March 29, 2002

Commissioner:

In connection with the filing of the enclosed National Stage application under 35 U.S.C. § 371, and before determination of the filing fee payable therein, please amend claim 32 to be as follows:

32. (Amended) Apparatus according to claim 25 in which the place of pipe use is a well drilling facility which includes a drilling operations platform.

REMARKS


The purpose and effect of this Preliminary Amendment is to cause claim 32 not to be a multiple dependent claim, and to eliminate 37 CFR § 1.16(d) from consideration in the determination of the filing fee payable for this § 371 application.

A version of claim 32 showing the before and after amendment content of the claim is attached.

Respectfully submitted,

CHRISTIE, PARKER & HALE, LLP

By


Hayden A. Carney
Reg. No. 22,653
626/795-9900

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VERSION TO SHOW CHANGES MADE

In the claims:

Amend Claim 32.

32. (Amended) Apparatus according to [~~any one of claims 25-31~~] claim 25 in which the place of pipe use is a well drilling facility which includes a drilling operations platform.

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HORIZONTAL DRILL PIPE RACKER AND DELIVERY SYSTEM

FIELD OF THE INVENTION

This invention pertains to structures and procedures for horizontally storing drill pipe adjacent to and preferably below the level of the drill floor of a well drilling installation. It also pertains to structures and procedures for moving stands of drill pipe between their storage locations and the drill floor.

BACKGROUND OF THE INVENTION

The quest for new oil reserves by the world oil industry forces the industry to seek oil and gas reserves in increasingly more demanding environments including the deep ocean. As the water depth for offshore drilling increases, the size of the equipment required to perform the drilling operations increases, as does the amount of subsea equipment required to extend the well bore to the surface of the ocean. Correspondingly, the costs of the equipment and of the drilling operation increase. A desirable way to offset the increased operating costs resulting from the use of current technology is to increase operating efficiency. An effective way to improve efficiency is to perform operations in less time, which translates into faster operating rates.

Drill pipe is one of the items affected by the increase in water depth. In the early years of offshore drilling, drilling operations were performed in water depths of a few hundred feet using five-inch (12.7 cm.) drill pipe weighing twenty and one-half pounds per linear foot (30.6 Kg./m.) including the connecting tool joints. Stands of drill pipe made up of three sections of such drill pipe, each nominally thirty-one feet (9.45 m.) in length, are called triples and weigh about one thousand nine hundred (1900) pounds (863.6 Kg). By comparison, triples for deep water drilling operations are made up of five inch (12.7 cm.) drill pipe weighing about 31 pounds per linear foot (46.3 Kg./m.), five and one-half inch (14.0 cm.) drill pipe weighing about 34 pounds per linear foot (50.8 Kg./m.), and six and five-eighths inch (16.8 cm.) drill pipe weighing up to forty-six pounds per foot (68.7 Kg./m.). The weight of drill pipe triples made up of these heavier pipes is about 2880 pounds (1309 Kg.) for the five inch drill pipe, about 3160 pounds (1436 Kg.) for the five and one-half inch drill pipe, and about 4300 pounds (1955 Kg.) for the six and five-eighths inch drill pipe.

The drill pipe used for deep water drilling is made from low alloy steel which has been heat-treated to high strengths. The material is stressed to high levels in use and, therefore, must be maintained free from significant scratches, gouges and other imperfections which can act as stress risers. To get the maximum life out of drill pipe, it must be protected from being scratched and gouged while it is being handled between a pipe storage location and the drill string where it is used. Drill pipe which is damaged beyond rigorous low damage limits must be discarded.

Horizontal pipe rackers commonly are used in floating offshore drilling rigs, especially those of ship-form configuration, because they contribute to the floating stability of the rig; they

1 lower the position in the rig of the stored drill pipe. Horizontal pipe rackers store drill pipe
triples in a horizontal position and include devices and mechanisms which transport the pipe, in
either direction, between the pipe racker and the drill floor. At the drill floor, drill pipe from the
racker is moved into a vertical position and is inserted into (connected to) the drill string.

5 Horizontal pipe rackers currently in use typically store several stands of drill pipe triples
in a single bin. When the pipe is put into and removed from the bin, it is rolled down an incline
to an indexing device that allows only one stand of pipe to be placed on a transporting device.
The rolling produces sliding and impact loading between adjacent drill pipe triples and between
drill pipe and the stationary pipe stops. While the pipe is stored in the bins, it can roll back and
10 forth in response to the vessel's own motions thus causing wear and damage between adjacent
drill pipe triples. Some pipe rackers prevent the drill pipe movement by storing each stand of pipe
in a separate locking mechanism, a tactic that generally limits the drill pipe to only one size. The
current method of transporting the drill pipe stands between the drill floor and the drill pipe
racker involves sliding the pipe along a long trough causing further wear and scratching along
15 the pipe where it contacts the trough.

United States patents 3,083,842 and 3,193,084 pertain to early pipe rackers, versions of
which remain in use.

It is seen, therefore, that a need exists for improvements in horizontal drill pipe racking
and handling systems to support deep water oil and gas drilling activities. Desirable aspects of
20 such improvements include increased horizontal pipe storage capacity, an ability to accommodate
differing diameters of drill pipe in the racker, an ability to store and to handle drill pipe stands
in ways which protect the pipe surfaces from being scratched, worn or gouged, and an ability to
rapidly and reliably move drill pipe stands between their storage locations and the drill floor.

25 SUMMARY OF THE INVENTION

This invention meaningfully addresses the needs noted above. It does so by providing
structural and procedural aspects of a horizontal drill pipe racking and handling system. The
system avoids the disadvantages and limitations of former horizontal pipe rackers and pipe
delivery arrangements. The system possesses the above-mentioned desirable attributes and
30 characteristics, as well as others.

The improved horizontal drill pipe racker provides positive control of the drill pipe stands
at all phases of handling. The drill pipe is safely secured while stored in a bin area. Positive
control of the drill pipe allows higher operating speeds to be achieved and reduces the time
required to transfer drill pipe between the horizontal pipe racker and the drill floor. The time
35 required to transfer drill pipe from the pipe racker to the drill floor is sufficiently low to keep
pace with the drill floor operations.

Generally speaking a pipe storage apparatus according to this invention comprises a pipe
storage bin, pipe support members for the bin, and drive mechanisms for the pipe support

1 members. The pipe support members are horizontally disposable in the bin at plural spaced
stations along the length of the bin. The pipe support members function to individually support
plural horizontal lengths of drill pipe in an array of plural vertically spaced layers and of plural
lengths of pipe in each layer. The pipe support drive mechanisms are selectively operable to
5 move the support members individually between deployed positions in which the support
members are in the array and retracted positions in which the support members are removed from
the array.

In terms of an overall system generally, the invention provides a drill pipe storage and
handling apparatus for a well drilling rig. A track extends from one end adjacent the drilling rig
10 to an opposite end remote from the rig. An elongate carriage is adapted to travel along the track
and to receive a length of drill pipe disposed longitudinally with respect to the track. A received
pipe length is supported on the carriage at spaced locations along the length of the pipe. A pipe
storage bin is disposed laterally of the remote end of the track. The bin includes horizontal pipe
support members which are cooperatively configured for individually supporting plural lengths
15 of drill pipe in an array of plural vertically spaced layers of pipe and plural lengths of pipe in each
layer. The pipe support members above the bottom layer are indexable between deployed
positions in and transversely of the array and retracted positions outside the array.

In terms of a general method for storing oil and gas well drill pipe, the method includes
the step of horizontally disposing a selected number of pipe lengths, as a first bottom layer
20 thereof, individually in upwardly open notches in the upper extent of a set of pipe supports
disposed transversely of the pipe lengths at stations spaced along the lengths. Another method
step is horizontally disposing further numbers of pipe lengths in further similarly notched pipe
support sets at each station atop the supports therebelow to create an array of plural layers of
plural lengths of pipe. A further step of the method is raising and lowering individual pipe
25 lengths directly from and to receiving notches in the pipe supports.

A method for storing, handling and moving drill pipe in association with a well drilling
rig having a drilling operations floor includes lifting a stand of drill pipe directly from an
individual storage position in an array of stand storage positions. Another step is placing the
lifted stand on a carriage adapted to support the stand at spaced locations along its length. The
30 carriage is moved towards the floor to place one end of the carriage at the floor. Another step
is elevating the one end of the placed stand above its placed positions on the carriage as the
carriage nears the floor. The stand is hoisted via its one end to a vertical position above the floor
while the other end of the stand is movably supported on the carriage.

35 DESCRIPTION OF THE DRAWINGS

The above-mentioned and other features and aspects of the structures and procedures
provided by this invention are described more fully below with reference to the accompanying
drawings in which:

1 FIG 1 is composed of FIGS 1A and 1B which overlap and respectively are elevation views of the rear (drill floor) and forward (horizontal racker) aspects of an overall pipe storage and handling system according to this invention as preferably provided on the main deck of a drillship;

5 FIG. 2 is composed of FIGS 2A and 2B which overlap and respectively are top plan views of the rear (drill floor) and forward (horizontal racker) aspects of the system shown in FIGS 1A and 1B;

FIG. 3 is a schematic top plan view of the arrangement of the major vertical structural columns which are components of the horizontal racker;

10 FIG. 4 is an enlarged top plan view of the forward and rear portions of the horizontal pipe racker which also is shown in FIG. 2B;

FIG. 5 is an elevation view of the forward end of the racker, the view taken aft of the forward buckboard;

15 FIG. 6 is an elevation view of the rear end of the racker, the view taken forward of the aft buckboard; in FIGs. 5 and 6, the bridge cranes are shown in positions different from FIG. 4;

FIG. 7 is a fragmentary elevation view of the forward portion of the racker as supported on the main deck of a drillship and shows, among other things, walkways and ladders for access of personnel to different portions of the racker;

20 FIG. 8 is a fragmentary top plan view of the pin cart and skate assembly which is a component of the pipe handling aspect of the system shown in FIGS 1 and 2;

FIG. 9 is a fragmentary elevation view of the pin cart and skate assembly shown in FIG. 8;

25 FIG. 10 is an enlarged fragmentary elevation view which shows the low lift aspects of the pin cart and skate assembly in the condition in which the pin cart is at its limit of travel along the skate assembly adjacent to the drilling floor;

FIG. 11 is a fragmentary top plan view which shows the pipe support sleeper and sleeper drive mechanisms associated with an outboard stanchion of the pipe racker, one sleeper being shown in a deployed position and the other being shown in a retracted position;

FIG. 11A is a simplified enlargement of a portion of FIG. 11;

30 FIG. 12 is a fragmentary elevation view showing all of the sleepers associated with an outboard pipe racker stanchion with the sleepers indexed to their retracted positions;

FIG. 13 is a simplified fragmentary elevation view of a drill pipe stand (shown in transverse cross-section) as supported between two vertically adjacent sleepers;

35 FIG. 14 is a simplified transverse cross-sectional elevation view of two vertically adjacent sleepers and shows the manner in which the sleepers can be keyed together in their deployed positions;

1 FIG. 15 is an elevation view looking forward along the path of movement of the skate cart at the drill floor of the drilling rig, and shows a pipe stabber and a pipe stand high lift mechanism which has its pipe engaging arm disposed in a retracted position;

5 FIG. 16 is a view similar to FIG. 15 and shows the pipe engaging arm of the high lift mechanism deployed into its working position;

10 FIG. 17 is a fragmentary side elevation view of the central portion of a racker bridge crane and the racker base below it in the vicinity of the center starboard column of the racker; FIG. 17 generally shows the mechanisms (somewhat different from those shown in FIGs. 1B, 7, 11, 11A and 12) for indexing the racker sleepers between their deployed and retracted (stowed) positions and for securing the crane lift columns and lift head assemblies into a stowed state;

15 FIG. 18 is an enlarged fragmentary elevation view of the sleeper indexing mechanisms according to FIG. 17 which are associated with the aft starboard main column of the racker;

FIG. 19 is a top plan view of the structure shown in FIG. 18;

20 FIG. 20 is a fragmentary top plan view of the hinged end of a sleeper base in the sleeper indexing arrangement shown in FIGs. 17-19;

FIG. 21 is a fragmentary side elevation view of a pipe stand lifting head and the adjacent supporting structure in the racker bridge crane shown in FIG. 17;

FIG. 22 is an enlarged end view of the lifting head shown in FIG. 21; and

FIG. 23 is an end view of the magnet assembly which is a component of the lifting head shown in FIGs. 21 and 22.

DESCRIPTION OF THE ILLUSTRATED EMBODIMENTS OF THE INVENTION

25 The following description and the accompanying drawings pertain to a presently preferred location of a preferred horizontal pipe racker in a drillship forward of a drill rig derrick in the drillship. Adjectives and other terms descriptive of horizontal directions (such as "forward," "aft," "port" and "starboard") are used with reference to that preferred location. It will be understood that the racker and its associated equipment could be located in other positions relative to the drill rig derrick in a different arrangement of drilling equipment, in which event other terms descriptive of horizontal directions would be apt and consistent with the scope and content of this invention.

30 In general terms, a pipe racker system includes a main foundation and support structure, multiple levels of indexable sleepers which define racks on which pipe is stored horizontally, a mechanism, such as a bridge crane, to transfer drill pipe stands between the sleepers and a drill pipe transporter, a drill pipe transporter to transport the pipe between the pipe racker and the drill floor, and a control system to provide the man-machine interface and to perform automatic control functions and to enable manual control operations to be performed. The drill pipe transporter preferably includes a carriage system which is comprised of a skate cart movable between the vicinity of the sleepers and the drill floor at the level of the floor, a further cart on

1 the skate cart for receiving and supporting the pin end of a pipe stand being moved to or from
the sleeper array. The drill pipe transporter also comprises low and high lift mechanisms for
elevating and lowering the box end of a pipe stand from and to the skate cart at the drill floor, and
a pipe stabber at the drill floor for guiding the lower end of a pipe stand between the carriage
5 system and the vertical axis at the drill floor at which drilling operations are performed.

More particularly, the structural aspects of a pipe storage and handling system according
to this invention includes a track which extends from one end adjacent the drilling rig to an
opposite end remote from rig floor. The skate cart travels along the track and protectively
supports a length of drill pipe disposed longitudinally with respect to the track. A pipe storage
10 bin is disposed laterally of the track. The bin includes the horizontal pipe support sleepers which
are cooperatively configured for individually supporting plural drill pipe lengths in an array of
plural vertically spaced layers of pipe lengths with plural lengths in each layer. The array of
sleepers is effective to support the pipe without subjecting any pipe length in the array to loads
due to pipes and sleepers above or adjacent to it in the array. The storage bin also includes drive
15 mechanisms connected to the pipe support sleepers. The drive mechanisms are selectively
operable to move the individual sleepers between positions in which the sleepers are in the array
and positions in which the sleepers are removed from the array when not supporting a drill pipe
length.

The system also includes a pipe lifter. The pipe lifter is disposable above the sleeper array
and is operable to move individual pipe lengths between the array and the carriage. More
particularly, the pipe lifter can include several controllable magnetic pipe lifting heads which are
spaced from each other along the length of the sleeper array. The several lifting heads are
effective to lift and to hold a pipe length. The lifter preferably also includes indexable
20 mechanical safety supports which are movable into and out of supportive relation to a pipe length
which has been lifted by the lifting heads. The pipe lifter preferably includes a bridge crane
above and movable between the skate cart track and the pipe storage bin. The crane includes
mechanisms for raising and lowering the several lifting heads and the safety supports if present
in association with the lifting heads.

The control subsystem preferably monitors the position of the lifting heads and permits
30 effective demagnetization of the lifting heads only when the load of a pipe length held by the
heads has been accepted by the carriage or by pipe support sleepers in the bin. The control
subsystem can be defined to operate at desired times and in desired sequences in a semiautomatic
manner.

Further, the system preferably includes a second pipe storage bin disposed on the other
35 side of the skate cart track from the first bin. A second pipe lifter preferably is provided for the
second bin. Where the system includes two pipe storage bins and two pipe lifters, the two pipe
lifters preferably are arranged to service either bin providing redundancy of pipe lifters.

1 The pipe storage and handling system also has procedural aspects which are described or
made apparent from the following descriptions of presently preferred and other embodiments of
the system.

FIGs. 1A and 1B and FIGs. 2A and 2B, respectively, are side elevation and top plan views
5 of a pipe storage and handling system 10 according to this invention. The system includes a
horizontal pipe racker subsystem 11 which is supported on the main deck 12 of a deepwater
drillship by a substructure 13; it is within the scope of this invention to locate racker 11 directly
on deck 12 or upon such other foundation as may be appropriate to the nature and location of the
operations supported by the racker and other associated structures described below. In the
10 preferred arrangement depicted, the racker substructure has a height on the order of twenty feet
(6.1 meters). Racker 11 is shown to be located in the drillship forward of a drilling rig 15 which
is supported on the vessel main deck by its own substructure 16 which locates a floor 17 of a
drilling platform a desired distance above the main deck. Drilling rig 15 is located over a vertical
passage 14 through the drillship hull. The racker substructure serves, among other things, to
15 elevate racker 11 above main deck 12 adequately that the path of fore-and-aft movement of a drill
pipe skate cart assembly 18 is horizontal and is substantially at the same height as the drill floor
above the main deck. The skate cart assembly provides a carriage which transports drill pipe
stands in a fore-and-aft direction within system 10. A skate truss 19, which supports the skate
cart in its movement, preferably along the vessel's longitudinal centerline, extends from the
20 forward end of the racker to the drill floor. As shown in FIG. 1A, a portion 21 of the skate truss
19 just forward of the drill rig can be removable to enable a transversely movable bridge crane
22 to be used to move a blowout preventer, or other equipment as needed, into position on the
vessel centerline before being moved rearward into alignment with a vertical well centerline 23,
i.e., the drilling axis as defined in the drilling rig. The removable portion of the skate truss, when
25 in its installed position shown in FIG. 1A, preferably is supported at its opposite ends on the
elevated athwartships rails and rail supports 16 provided for crane 22.

As is characteristic in drillships, the drilling rig 15 is located at about amidships of the
vessel hull over the vertical passage 14 (known as a "moonpool") through the hull. The rig
includes a derrick (not shown) equipped with a crown block atop the derrick, a traveling block
30 in the derrick, and a drawworks operable for raising and lowering the traveling block along well
centerline 23 above the rig floor. The drawworks also can be operated to raise and lower other
blocks or lifting hooks in the derrick, and to perform other functions as known in the oil and gas
drilling industry.

Skate cart 18 and skate truss 19 are components of a pipe handling subsystem in the
35 overall pipe storage and handling system 10.

As shown best in FIGs. 1B and 3-6, the structure of racker 11 includes an assembly 26
composed of a relatively massive base 24 and of columns 25. Assembly 26 preferably is
constructed to be stiff and rigid in its own right. The substructure 13 which supports that

1 assembly on the vessel main deck both supports and secures the assembly in place as a part of
the ship structure and substantially isolates that assembly from bending deflections in the ship
hull. There preferably are nine columns which are denominated generally as 25 in the drawings
except in FIG. 3 which is a plan view of the column pattern in which the columns have positions
5 A, B, C, D, E, F, G, H and J. Positions A, B and C, positions D, E, and F, and positions G, H and
J are each arrayed along lines transverse to the vessel centerline respectively at the rear, center
and forward positions of the racker. Those columns at positions A, D, and G are located on a
line to port of and parallel to the centerline, those at positions B, E, and H are on the centerline,
and those at positions C, F and J are on a line to starboard of and parallel to the centerline. The
10 columns at the corner positions A, G, C and J are of the same height which is greater than the
height of the centerline columns and the outboard center columns D and F. The upper ends of
the outboard column pairs at the forward and aft portions of the racker are interconnected by
transverse beams 27 which have outboard extensions from those columns, as shown in FIGs. 5
and 6. The space below the upper ends of the centerline columns and between those columns
15 and the port outboard columns comprises a port pipe storage bay or bin 28. A starboard pipe
storage bay or bin 29 is formed below the tops of the centerline columns and between those
columns and the starboard outboard columns. The centerline columns support corresponding
portions of skate truss 19. Columns 25 are interconnected at their bases by longitudinal and
transverse girders which are components of base 24.

20 The upper ends of the transversely aligned outboard corner columns and their
interconnecting beams 27 support transversely oriented forward and aft guide and support rails
31 for a pair of longitudinally extending port and starboard bridge cranes 32 and 33, respectively.

Additional vertical members are located in the racker along the outboard sides of storage
bins 28 and 29, as at 34 in FIG. 1B. A transverse vertical plate 35, attached to structural
25 members, is located at each of the ends of the bins and supports wooden buckboards 36 which
face toward the respective bins. The buckboards limit fore and aft sliding motion of pipe stands
which are individually supported in the racker without damaging the threaded connection features
at the ends of the stands.

30 The cross-hatched area in FIG. 7 represents the vertical and forward limits of the volume
in the racker in which a triple stand of drill pipe can be found either when stored in the racker or
when being moved in a horizontal attitude within, to or from the racker.

As noted above, the racker structure includes means for individually supporting each triple
stand of drill pipe which is to be stored at any time in the racker. Those means include a plurality
of horizontal sleepers 38 and 39 which cooperate with each other and with the stands in the
35 manner shown in FIG. 13. Sleepers 38 are movable, whereas bottom sleepers 39 are fixed and
are carried by the transverse portions of the racker base 24 as shown in FIG. 6. The movable
sleepers are sufficiently long that, when in their deployed positions, they extend across the entire
width of the respective pipe storage bin between transversely aligned ones of columns 25. The

1 pipe stands are stored in an array of layers in each of bins 28 and 29, there being plural stands in
each layer. Except for the stands in the topmost layer, each stand is disposed between two
vertically adjacent sleepers. The vertically adjacent sleepers are configured to keep adjacent
stands in a layer in spaced relation to each other, and to carry the vertical loads of sleepers and
5 of stands supported by the sleepers to the racker base 24 within the sleepers themselves and not
through any stands in the storage array. The arrows in FIG. 13 represent the way in which
vertical forces are carried in the deployed sleepers around the pipe stands received in the sleepers.
Thus, the only forces applied to a stand when stored in the racker are the forces attributable to
that stand itself due to its own weight and due to motions of the drillship. A given stand, when
10 stored in the racker, does not have applied to it any portion of such forces associated with any
other stand. In the preferred racker shown in the drawings, sleepers 38 and 39 are provided at
each of three stations spaced along the length of each pipe storage bin.

15 The fixed sleepers 39 are notched at intervals along their top surfaces. The movable
sleepers 38 are similarly notched at the same intervals along the top and bottom surfaces. The
upwardly opening 41 notches have sloping sides; the downwardly opening notches 42 can be
accurately shaped. The cooperating notches in vertically adjacent sleepers are sized so that a pipe
stand 43 disposed in the space formed by a notch 41 and a notch 42 contacts only the upwardly
opening notch 41. Therefore, the stand is not subjected to any forces attributable to sleepers or
stands above it in the bin in which the stand is stored. If desired, the sloped surface of upwardly
opening notches 41 can be defined by wood or by pieces of other materials which are softer than
20 the metal of a pipe stand and which preferably cannot participate in electrolytic corrosion
processes with the pipe. Each cooperating pair of notches 41, 42 has sufficient open area that the
sleeper array can accept pipe of either a specified diameter or with a defined range of diameters
with variations from truly straight to in excess of 0.2 percent of pipe length deviation from truly
straight.
25

The stacked sleepers are mechanically keyed and interlocked between adjoining sleepers
to prevent one sleeper from deflecting sideways relative to the long dimension of the sleeper; see
FIG. 14. This assures the sleeper stack will remain aligned one sleeper above the other and that
the drill pipe loads will follow a path down through the stacked sleepers to the base crossbeam
30 supporting it. The notches 41, 42 in the sleepers preferably are configured to allow the notches
to accommodate drill pipe ranging in diameter from five (5) inches to six and five-eighths (6 5/8)
inches; alternatively, the sleepers can be notched to accommodate a given size of pipe, thereby
maximizing the number of stands disposable in a layer of the storage array, and the sleepers can
be disconnectable from their indexing drive mechanisms and supporting bases to enable one
35 sleeper to be exchanged for another of different capacity. The sleepers cooperate with pipe
stands at locations along the stands between the pin and box features at the opposite ends of each
of the individual pipe lengths forming the stands.

1 Resting the drill pipe between the sloped sides of a notch 41 increases the contact force
between the sleeper and the drill pipe. The sliding friction between the pipe and the sleeper is
proportional to the contact force and, thus, the sliding friction between the sleeper and the drill
pipe is increased. The increased friction keeps the pipe from sliding along its length when it is
5 subjected to dynamic loads along its length due to ship motions. When the drill pipe is subjected
to dynamic loads in the direction of the sleeper length, the pipe is wedged into the sloped sides
of the sleeper notch and will not roll in the sleeper.

The sleepers are mounted to be indexed to lift them off the sleeper below and to rotate
them out of the pipe bin to expose the next layer of drill pipe stands. Conversely, when a sleeper
10 becomes filled with a drill pipe stand in each notch 41, the next sleeper above is rotated to a
position over the stack of sleepers below and then lowered until it rests on the topmost sleeper
below. That mechanically locks the two sleepers together and contains the layer of drill pipe
below the sleeper between the sleeper supporting the pipe and the one placed over it.

Drill pipe stands are moved between the sleepers in the pipe bins and the pipe transporter
15 (i.e., skate cart 18) by means of a special bridge crane illustrated in FIGs. 1B, 2B, 4, 5 and 6.
Preferably, there are two bridge cranes 32 and 33; while a single bridge crane could be provided
to service both pipe storage bays, a pair of bridge cranes is preferred for efficiency and for
redundancy if needed. Each bridge crane spans the distance between the two crane rails 31 at the
ends of the pipe racker. Self-propelled trucks 44 at the ends of the bridge crane run on the rails
20 31 and drive the crane from one side of the pipe racker to the other. The rails 31 preferably are
channel-like in cross-section, and the crane truck wheels preferably run in the channels so that
the trucks are captive to the rails; see FIG. 6, e.g. The trucks of each bridge crane are
synchronized to make them move together in unison. A set of two or more (preferably three)
vertical lift members (columns) 45 are mounted in the crane bridge and are constrained to move
25 only up and down relative to the bridge. The lift columns in each bridge crane are synchronized
to move together and are connected at the lower ends by a long beam identified as a strongback
46, also called a spreader bar. The strongback is a carrier for permanent magnet lift heads 47
which function to attach to a stand of pipe and to release it when the magnetic field is broken.

The magnetic lift heads which make contact with the drill pipe are in the form of two long
30 permanent magnet bars and preferably are arranged in a manner that they form an upside down
trough. While the trough can be "V" shaped (see FIG 23), it is preferred that the trough be
semicircular, most preferably with a radius which fits the pipe being handled. The trough lays
parallel to the direction of the drill pipe. The magnets preferably are attached to the strongback
in a way that the they can rotate and translate in small amounts to allow the them to align
35 themselves with the drill pipe being picked up. Aligning the magnetic heads with the drill pipe
greatly increases the contact area between the pipe wall and the magnetic heads and hence
maximizes the holding capacity of each magnetic pipe lifter. The pipe stand lift heads are
described more fully below with particular reference to FIGs. 21-23.

1 When moving drill pipe from the pipe bins to the drill pipe transporter, a chosen one of
bridge cranes 32, 33 moves sideways on its bridge trucks from a transfer position over the
transporter to a discrete position over the drill pipe stand to be transferred; while both cranes can
be operated concurrently, it presently is preferred to operate only one crane at a given time. The
5 strongback is lowered by the vertical lift members 45 until the magnetic lift heads 47 are in
contact with the drill pipe. The magnetic fields generated by the lift heads are effective to attach
the lift heads to the drill pipe. The lift members are then raised to their highest positions in the
bridge crane with the drill pipe coupled to the strongback by the magnets. The bridge crane
moves from its position over the pipe bin to a position directly over the drill pipe transporter
10 which is located along the center of the racker below the transfer position of the crane. The
strongback is lowered until the drill pipe rests on the transporter skate cart, with the pin end of
the pipe stand engaged in the pin cart; only then the magnetic fields are killed to release the drill
pipe. After the magnets release the drill pipe, the bridge crane raises the strongback to its full
upper position and then moves to pick up the next stand of drill pipe from a pipe storage bin.

15 Moving drill pipe from the drill pipe transporter to a pipe bin is the reverse of moving the
pipe from the pipe bin to the transporter. The bridge crane picks up the drill pipe from the drill
pipe transporter at the transfer position of the transporter carriage. The crane moves it to a
specified open slot (position) in a pipe bin sleeper set. The pipe stand is released from the lifting
heads after the weight of the stand has been accepted by the pipe support sleepers. When a
20 bridge crane is operating semi-automatically, the bridge crane will always stop at its transfer
position above the transporter. The pipe is lowered to and released on the transporter only by
direct manual command from a control console. Similarly, when the strongback is waiting over
the transporter to pick up a drill pipe, it will commence lowering and attaching to the drill pipe
only when it is commanded to do so from the control console. This feature helps avoid several
25 hazards that otherwise would have to have sophisticated interlocks to prevent. The drill pipe will
not be lowered and released when the transporter is not in position to receive it. The drill pipe
will not be lowered or released while the transporter is still moving. Adherence to those rules
simplifies the racker system controls and timing since the bridge crane always starts a cycle from
a "hold" at its transfer position and ends the cycle at the same position. It does not have to be
30 timed to meet the drill pipe transporter.

The drill pipe transporter, (see FIGs. 8, 9 and 10) moves the drill pipe between the
horizontal pipe racker where the drill pipe is stored and the drill floor where the pipe is used. The
transporter pipe carriage arrangement preferably is comprised of the long skate cart 18, a second
cart-like device identified as a pin cart 49, and drive systems (not shown in FIGs. 8, 9, and 10)
35 to move the skate and pin carts. The pin cart is carried by and is driven along the skate cart.

The skate cart preferably is about 88 feet (26.8m) long and the drill pipe stand being
transported will range from about 91 feet (27.7m) long to about 95.5 feet (29.1m) long. The drill
pipe stand being transported on the skate cart will extend at its rear or box end beyond the rear

1 end of the skate cart from three feet (.9m) to eight feet (2.4m). The remaining length of the drill
pipe will be cradled on the skate cart, and the forward pin end of the pipe stand will be carried
in pin cart 49. The skate cart rides on wheels that run captively in channels affixed to the skate
truss. The channels are installed on the skate truss 19 and run from the end of the pipe racker
5 farthest from the drill floor to the drill floor.

The pin cart rides on wheels which captively engage tracks mounted on the skate cart, thus
allowing the pin cart to move relative to the skate cart. When drill pipe is loaded on the skate
cart, the pin (forward) end of the pipe rests in the pin cart while the box (rear) end of the drill
pipe extends over the other end of the skate cart. When the skate cart approaches the drill floor,
10 the box end of the drill pipe is raised by a low-lift pipe lifter 52 and is held about four to five feet
above the drill floor. An elevator, typically carried by the traveling block in the drill rig derrick,
is attached to the box end of the drill pipe so held above the drill floor, and the box end of the
drill pipe is lifted up above the drill floor by the drawworks and traveling block. As the box end
of the pipe is lifted, the skate cart will continue to travel toward the drill floor until it reaches its
15 rearward limit. Then the pin cart holding and supporting the pin end of the drill pipe will move
along the skate cart toward the drill floor with the pin end of the pipe as the pipe is raised and
rotated from a horizontal to a vertical position in the derrick. When the drill pipe becomes nearly
vertical, it is lifted off the pin cart and installed in the working drill string; before the drill pipe
is lifted off the pin cart and until it has been made part of the working drill string, it is held in the
20 head 72 of pipe stabber 73. The pin cart is moved back to the other end of the skate cart,
preferably as the skate cart is returned to its position forward in the pipe racker.

FIGs. 5 and 6 show in elevation views the port and starboard pipe storage bins. In each
of those FIGs., one bin is shown full and the other is shown empty with all movable sleepers 38
removed from the bin volume. As shown in FIG. 11, each sleeper 38 has a deployed position in
25 which it extends from its supporting outboard column 25 to the adjacent centerline column. Each
sleeper also has a retracted position which is disposed at a 90° relation to its deployed position
and in which the sleeper is outside the outboard side of the bin as defined by the inboard faces
of the outboard columns; see, e.g., FIGS. 1B, 4, 7, 11, 12, 17 and 18.

As shown in FIG. 5, e.g., there can be 18 movable sleepers 38 in each bin in association
30 with each outboard racker column 25; fewer or more sleepers can be provided, as desired. That
set of 18 sleepers is composed of a group of 9 even numbered sleepers and a group of 9 odd
numbered sleepers. For purposes of example, the odd numbered sleepers in the set (counting
upwardly) are associated with the aft face of the adjacent outboard column, while the even
numbered sleepers in that set are associated with the forward face of that same column. (It will
35 be noted that FIG. 12 shows 19 movable sleepers associated with a racker column.)

As shown in FIG. 11, each sleeper 38 is connected to a base 54 which preferably is L-
shaped in plan view. Each base has a major leg defined by an elongate frame 55 which is parallel
to the related sleeper. Each base also has an arm 56 which extends laterally from one end of the

1 frame and to which one end of the sleeper is connected. The end of frame 55 opposite from arm
56 defines a vertically oriented sleeve-like hub 57 which encircles two vertical bearing tubes 58
in one form of sleeper mounting and indexing arrangement of this invention; see FIG. 11A. The
bearing tubes 58 are vertically in line with each other, one being at the bottom of the hub 57, the
5 other at the top of the hub 57. The two bearing tubes 58 are separated by a distance of about 3
inches. The separation between the bearing tubes 58 provides an opening through which a fork
62 can move to engage with or disengage from a vertical, notched shaft 60 in FIG. 11A.
Adjacent each of those openings between the bearing tubes 58 in each sleeper hub, shaft 60 is
notched in parallel diametrically opposed locations; the notches form aligned horizontal keyways
10 in the shaft, as at 63 in FIG. 11A. The vertical shaft 60 preferably is of sufficiently long extent
that it is common to all the sleepers in a group; there are two shafts 60 associated with each
outboard column 25, one for each sleeper group at that column. Each shaft 60 is axially and
angularly movably mounted to the column 25 via support brackets 59 spaced along the vertical
length of the shaft 60 between vertically adjacent sleepers. Each bracket 59 is comprised of a
15 structure attached to the column on one end and attached to a vertically oriented sleeve-like hub
and a tube bearing similar to the sleeper hub 57. The vertical shaft 60 is free to move vertically
and to rotate in all the support brackets 59, and in the sleeper hubs 57 except when a selected fork
62 is engaged in a corresponding pair of aligned notches on the shaft. In that case, the sleeper
associated with the engaged fork will move angularly and vertically with the shaft 60. Axial and
20 angular motions of each shaft are produced by suitable drive mechanisms 53 (see FIG. 1B) which
can be located along side the racker base 24 and to which the lower end of the shaft extends.

As shown in FIGs. 11 and 11A, each sleeper base frame preferably carries a double acting
ram 61, preferably a pneumatic ram, which is coupled adjacent the base hub to a horizontally
slidable fork 62 which opens toward shaft 60. The fork is movable horizontally through the
25 opening between the bearing tubes 58 in the adjacent hub 57 into engagement of its two legs in
the keyway notches 63 in the shaft. Engagement of a fork with the shaft connects the
corresponding sleeper and its base to the shaft so that the sleeper can be moved vertically and
angularly by axial and angular motion of the shaft. Each fork can be fully disengaged from the
shaft 60.

30 Reference is made to sleeper 38 of FIG. 11 which lies in the bottom left portion of FIG.
11. That sleeper is in its retracted position and is supported about midway along its length in a
holder and support bracket 64 carried by a racker side member 34. That sleeper is to be moved
to its deployed position in the adjacent pipe storage bin; the other sleeper depicted in FIG. 11 is
shown in its deployed position. Movement of a sleeper from its retracted position to its deployed
35 position is achieved by the following operations:

1. ram 61 is operated to move fork 62 into engagement with the keyway notches 63
in shaft 60;
2. shaft 60 is raised to lift the sleeper out of engagement with its holder 64;

- 1 3. shaft 60 is rotated 90° in a chosen direction, thereby swinging the sleeper about its hinge axis at the shaft from its raised and retracted position to a raised and deployed position;
4. shaft 60 is lowered to cause the sleeper to register with the deployed sleeper below it and to cause the chevron or tongue-and-groove contours on the abutting sleeper faces (see FIG. 14) to engage;
- 5 5. ram 61 is operated to disengage the sleeper's fork from shaft 60; and
6. shaft 60 is rotated 90° in a direction opposite to the chosen direction to return to its starting angular position where it is ready to be engaged later by the fork of the
- 10 next highest sleeper along that shaft.

Obviously, movement of a deployed sleeper to its retracted position involves essentially the reverse of the operational sequence described above.

15 The control arrangement for system 10 preferably includes sensors and interlocks which monitor the retracted/deployed status of each sleeper in each pipe storage array, and which assure that only the proper sleepers are engaged with and moved by shafts 60 at any time during loading or unloading of drill pipe into or from each pipe storage bins. If, for some reason, one or more sleepers in a set fails to deploy or to retract when commanded, the operator is alerted and a signal is provided to the control system to prevent the crane in service from lowering pipe until the sleeper positioning issue has been addressed. It is preferred to index the set of sleepers for a

20 given layer of a bin in unison.

FIG. 11 shows the benefit of mounting each movable sleeper 38 to an L-shaped base 54. That geometry of the sleeper base enables the hinge structures, about which the bases swing as the sleepers move between their deployed and retracted positions, to be located adjacent to the outboard corners of the outboard columns 25 of the racker. When the sleepers are retracted, they are located outside the adjacent pipe storage bin. When they are deployed, the L-shaped sleeper bases wrap around the adjacent column so that the sleepers extend transversely across the bin and span the space between the corresponding outboard column and the related center column.

25 FIG. 11 also shows that a brace 65 can be connected from the unhinged end of each sleeper base frame to a point on the sleeper which is near its midlength. Braces 65 can lie in planes between the upper and lower surfaces of the sleepers and so will not interfere with the drill pipe lengths racked on the deployed sleepers.

30 FIGs. 5 and 12, e.g., show how the odd and even numbered sleepers in each sleeper set interdigitate in their deployed positions.

35 Each bridge crane lift column 45 preferably is driven vertically, in one method, by a pair of drive motors 67 (see FIG. 4, e.g.) which drive pinions meshed with a respective one of a pair of racks which are carried along opposite sides of the lift columns. Alternately, the lift columns maybe raised and lowered by a single shaft having three pinions to engage the corresponding racks on the three lift columns.

1 As noted above, it is desirable that the control subsystem of system 10 receives
information about the current position of each bridge crane 32, 33 and about the vertical position
of the pipe lifting heads 47 carried by each bridge crane. It is also important that all lift columns
on a bridge crane be operated in close synchronism, and that the self-propelled trucks 44 at the
5 opposite ends of each bridge crane be operated in close synchronism. Synchronizing and position
informative signals preferably are generated by each lift column drive mechanism and by each
bridge crane truck as they are operated. Those signals can be generated in similar ways. For
example, as a bridge crane traverses racker 11 on its support rails 31, it can operate switches
which are spaced along each track. Also, as each crane truck moves, rotation of one of its drive
10 shafts can operate an encoder. The encoder output signals generated by the trucks at the opposite
ends of the crane can be compared and the result used to synchronize the operation of the trucks.
Operation of the switches along the crane rails can be used to periodically reset the encoders.
The combination of switch signals and encoder signals can provide high precision information
about the location transversely of the racker of the pipe lifter heads carried by the crane. Similar
15 switches and encoders can be used in the lift column drives to synchronize the motions of the
several columns in each crane and to provide high-precision information about the vertical
position of the crane's pipe lifter heads.

Switches operated by the sleepers can indicate whether a sleeper is deployed or retracted.
Combining the disposition of the sleepers and their known height with the vertical position of
20 the lift columns is sufficient information to accurately position the vertical lift columns and the
magnetic heads to pick up or set down pipe in the sleepers. Also, it is preferred that the sleepers
have associated with them sensors which are effective to detect the presence of personnel on the
sleepers or on the top pipe stands supported by the sleepers.

FIG. 6 shows that the bridge cranes 32, 33 can have stowed positions outboard of the pipe
25 storage bins on the positions of rails 31 which extend laterally away from the pipe storage bins.
When the cranes are stowed, the lift columns for the cranes can be lowered to essentially fully
depend from the crane truss (bridge) structures and trucks. In that condition, the crane lift
columns and the structures carried by the lower ends of those columns can be secured from
pendulous motion by being connected to keeper brackets 68 which preferably extend outwardly
30 from the racker sides. This storage ability for the bridge cranes and their lift columns is
especially useful in the context of use of racker 10 on a floating vessel. Weather or sea state
conditions may make it prudent and desirable to discontinue drilling operations for a time.
During that time, the cranes are stowed as described and their lift columns are secured from
pendulous motions which otherwise could result in damage to them or to other components of
35 the racker.

FIGs. 8 and 9 respectively are top plan and side elevation views of skate cart 18. The
skate cart has an elongate horizontal, rather narrow truss-like frame 70 shown in FIG. 8. At
spaced locations along its sides, frame 70 is supported by wheels which run captively in tracks

1 which extend along the top of the skate truss 19. The skate cart is drivable back and forth along
the skate truss by a drive motor 71, preferably located under the cart rails near the rear end of
racker 11 at a location which is under the cart when the cart is either at its forward limit of travel
or at its rearward limit of travel. The motor preferably is an electric motor which rotates a pinion
5 which meshes with a rack carried by frame 70. The skate cart drive preferably includes a brake
which is operable to stop the cart and to hold it at any position within its range of travel. Also,
it is preferred that horizontally disposed shock absorbers are carried by fixed structures at the
opposite limits of skate cart motion to cooperate with the skate cart as it moves back and forth
during operation of system 10.

10 The forward limit of travel of the skate cart is shown in FIG. 1B. The rearward limit of
travel of the skate cart is within the perimeter of drilling rig floor 17 toward well centerline 23
from the forward edge of the rig floor and from a horizontal pipe stop 130 shown in FIG. 17. As
is known in the oil and gas drilling industry, the pipe stop is a horizontal bar which is located
about four to five feet or so above the rig floor adjacent the forward limit of travel of a
15 horizontally reciprocable pipe handling head 72 of a pipe stabber 73; see FIGs. 1A and 2A. The
pipe stop limits the rearward motion of the pin end of a pipe stand 43 which has its box end
engaged in a pipe elevator and raised in the rig derrick above the rig floor in the course of moving
the pipe stand from racker 11 into connection with the upper end of a pipe string supported along
the well centerline substantially at or above the rig floor. The pipe stop comes into effect when
20 the pin cart 49 on the skate cart also is at its rearward limit of movement along the skate cart.
The pipe stop holds the lower end of the essentially vertical pipe stand in a position for
engagement of the stand, adjacent its lower end, by the gripper mechanism which is part of the
pipe stabber head 72. When the stand has been engaged by the pipe stabber, the stand is raised
so that the pin end clears the pipe stop. The stabber then is operated to move the lower end of
25 the pendulously supported pipe stand along a fixed path into alignment with the well centerline
where it can be connected to the upper end of the pipe string.

Pin cart 49 is a small carriage which is captive to and moves in tracks atop and along the
length of the skate cart. The skate cart is shown in FIGs. 2B and 9 at its forward limit of travel
along the skate cart; the skate cart also is at its forward limit of travel. The pin cart is shown in
30 FIG. 10 at its rearward limit of travel along the skate cart within the lengthwise extent of low lift
mechanism 52. The pin cart is driven along the skate cart by a drive motor 75 which preferably
is located at the forward end of the skate cart and preferably is an electric motor. Motor 75
preferably is coupled to the pin cart by a cable loop which is arranged for powered movement of
the pin cart in either direction along the skate cart. The pin cart defines a rearwardly and
35 upwardly open receptacle into which the pin end of pipe stand 43 can fit. The pin cart is capable
of accepting a substantial vertical load from a pipe stand as the stand moves from a horizontal
position on the skate to a nearly vertical position in the derrick, or vice versa. The pin cart drive
preferably includes a brake and an overrunning clutch which permits the pin cart to be moved,

1 by a pipe stand engaged with the cart, at a speed different from the cart speed corresponding to the operation of the pin cart drive motor.

At its extreme rear end, the skate cart carries low lift mechanism 52. The low lift mechanism is provided to raise the rear box end of a pipe stand from the skate cart, as the skate
5 cart approaches the rig floor, an amount which is adequate to place the stand box end above the pipe stop adequately that the stand box end can be reached by elevators for lifting into the rig derrick. The box end of the stand is then engaged by the pipe elevators which are supported by the rig's traveling block, e.g. The traveling block and the elevators are used to raise the stand box end into the derrick as the stand's pin end is moved by the pin cart, or by the pipe stand itself,
10 further toward the rig floor. The low lift mechanism is retracted (lowered) when the box end of the stand has been lifted from it, so that the rear end of the low lift mechanism can pass beneath the pipe stop toward the center of the rig floor (see FIG. 1A).

The presently preferred low lift mechanism preferably is pneumatically powered and preferably includes a pair of pneumatic rams 76; see FIG. 10, e.g. The rams 76 are mounted to the skate cart sides on opposite sides of the pin cart path of movement. Each ram is connected
15 to a crank arm 79 which is coupled to a corresponding one of two lift arms 77 so that extension and retraction of the rams causes the rearmost ends of the lift arms to raise and to lower. The preferred coupling between crank arms 79 and lift arms 77 are meshed gear sectors carried by each of the arms concentric to the pivot axes of the arms. The rear ends of the lift arms mount
20 between them a pipe support roller 78, rotatable about a horizontal axis, which preferably has a linearly tapered hourglass configuration. The lowered position of roller 78 is defined to support a pipe stand on the skate cart at a location on the stand adjacent the stand box end. As the support roller is raised to lift the stand box end from the skate cart, its profile acts to keep the pipe stand positioned along the centerline of the skate cart. The low lift mechanism is operable
25 in a reverse manner to lower to the skate cart the box end of a pipe stand being transferred from the derrick to the pipe racker for storage.

As shown in FIG. 8, e.g., the structure of the low lift mechanism forward of roller 78, i.e., between the roller and rams 76, is arranged on the opposite sides of the skate cart clear of the pin
30 cart's path of movement. As a result, the pin cart 49 can move within the length of the low lift mechanism to the extreme rear end of the skate cart. The pin cart can be positioned within the perimeter of the rig floor closely adjacent to pipe stop 130 when it and the skate cart are at their rearward limits of travel.

In addition to the pipe stabber 73 at the forward margin of the rig floor 17 to handle a pipe stand moving between the rig floor and pipe racker 11, a high lift mechanism 80 also is present
35 there, preferably attached to the pipe stabber. Also there, adjacent to the pipe stabber, and also on the opposite side of the path along which the pipe stand moves, is a pipe guard 132 to contain pipe in the pipe path. See FIGs. 15 and 16. The high lift 80 can be used to advantage during drilling operations. During drilling operations, the drill pipe string can be held by a top drive unit

1 which rotates the drill string and which holds the drill string at an essentially constant load at the
drill bit by means of vertical motion compensation equipment known to the offshore drilling
industry. This constant bit load can be maintained even as the ship heaves up and down in the
5 seas. When it becomes necessary to add more pipe to the drill string, the drill pipe is seated into
slips in a rotary table and disconnected from the top drive. The top drive is raised to the top of
the derrick with the next stand of pipe in its elevators. When the new stand of pipe is vertical,
it is lowered into the box end tool joint at the top of the drill string, whereupon drilling may
continue. In order to prevent the drill bit from striking the bottom of the well while it is
10 suspended in the slips without motion compensation, the pipe string is raised about 15 feet above
the drill floor before being set in slips. The high lift can be used to elevate the drill pipe box end
from the low lift level to a greater height where the top drive elevators can reach it.

The high lift has a carriage 135 which is raised and lowered preferably by a pneumatically
driven chain. FIG. 15 shows carriage 135 in upper and intermediate positions while FIG. 16
15 shows the carriage at upper, intermediate and lower positions. The carriage can have an arm 82
which will articulate from a vertical, upright position down to a horizontal position, and a
pneumatic ram 136 to rotate the arm from one position to the other. The arm is connected in a
manner that a pipe striking the arm from below will raise the arm. When a pipe is lowered onto
the arm from above, the arm is stopped in the horizontal position and holds the pipe in the cradle
on the arm.

20 The high lift is normally stowed in the upmost position with the arm rotated to the vertical
position out of the way of pipe being tripped in or out of the well. When the full length of the
drill string is in the well and drilling, the high lift carriage is lowered to the lowest most position
and the arm 82 is rotated to the horizontal position where it remains until the pipe skate cart is
moved aft with the next stand of drill pipe.

25 FIGs. 1, 2, and 7 illustrate the provision of walkways 84 and ladders 85 on the racker to
afford access of personnel to it.

A second form of the sleeper indexing arrangement, somewhat different from the
arrangement described above with reference to FIGs. 11 and 11A, is illustrated in FIGs. 17-20
and is presently more preferred. A vertical shaft 60 is associated with each group of sleepers 38
30 at each sleeper station of each pipe storage bay. Each shaft 60 is located adjacent the forward or
aft outboard corner of a racker column 25, as appropriate. Each shaft 60 is supported for rotation
and axial movement in a series of bearings 90 which are spaced along the shaft and are mounted
in brackets 91 (see FIG. 20) affixed to the adjacent racker columns 25. Each bracket 91 and
bearing 90 is located below the downward limit of travel of a respective sleeper base 54 and
35 defines that limit. Just above each such bracket 91, the shaft is transversely notched at
diametrically opposed locations to define a pair of horizontal keyway slots 63. A fork 62 is
mounted for slidable movement within the sleeper base. The sleeper base has a terminal hub 57,
which surrounds the shaft and mounts the base to the shaft for rotation about and axial movement

1 along the shaft. The fork is reciprocated in the sleeper base by a double-acting ram 61. The fork
thus is moveable by the ram 61 into and out of engagement with the vertical shaft 60 at the
keyway slots 63. Engagement of a fork 62 in keyway slots 63 of a shaft 60 releasably connects
the corresponding sleeper base and hub to the shaft so that the base and hub follow axial and
5 angular motion of the shaft.

Each vertical sleeper indexing drive shaft 60 is rotatably supported vertically at its bottom
end on a pedestal or platform 92 defined at the upper end of the piston 93 of a vertically disposed
ram 94 located below the shaft; see FIG. 18. That ram is operated to raise and to lower the
corresponding shaft. Also, there is a shaft angular drive assembly 95 coupled to each vertical
10 shaft above the corresponding vertical drive ram. Each assembly 95 preferably includes a crank
arm 96 affixed to the shaft and a double-acting horizontally disposed ram 97 pinned between the
end of the crank arm and the racker base 24. The mounting of each horizontal ram 97 between
its shaft crank arm and the racker base is arranged to enable the angular drive assembly to
accommodate and to follow vertical motion of the shaft by an amount which is adequate to lift
15 a stowed sleeper 38 out of engagement with a sleeper support 64 and adequate to lift a deployed
sleeper out of engagement with a deployed sleeper below it.

FIG. 19 shows that the horizontal drive ram 97 for the aftmost shaft associated with a pipe
storage bay preferably is located adjacent the rear end of the racker base. The forwardmost
horizontal drive ram associated with a pipe storage bay preferably is similarly located relative
20 to the front end of the racker base. All other horizontal drive rams are located adjacent the sides
of the racker base.

The operation of the vertical drive and angular drive mechanisms shown in FIGs. 17-19
is according to the six-step sequence described above.

FIG. 17 shows a presently preferred arrangement for securing the lower end of a bridge
25 crane lift column 45 from movement when the lift column is stowed when at its lowest position
in the bridge crane. A dummy section 98 of drill pipe is supported in a horizontal fore-and-aft
position outboard of the adjacent vertical 94 and angular 95 drives for a sleeper indexing drive
shaft 60. The pipe dummy section is located vertically below the place occupied by a
corresponding magnetic lifting head when a bridge crane is in its stowed position as described
30 above. The dummy pipe section is carried on a support bracket 99 mounted to the outer face of
the racker base. The magnetic lift head 47 carried by the lift column can make contact with the
top of the dummy pipe section, and a mechanical safety latch 105 associated with the magnetic
lift head can be engaged around and below the dummy pipe section. Such forms of contact with
the dummy pipe section cooperate to lock the lower end of the lift column to the dummy pipe
35 section. A drill pipe dummy section preferably is provided for each lifting head in each bridge
crane at spaced locations along the outboard sides of the racker base.

FIGs. 21-23 show details of a presently preferred magnetic lift unit or head 47, several
of which are carried by each racker bridge crane spreader bar 46 at spaced locations along the bar.

1 The central part of each lift head is occupied by an elongate magnet assembly 101 (see FIGs. 21
and 23) which includes a pair of slab-like permanent magnets 102. The magnets are in spaced
parallel relation to each other and are in planes parallel to the length of the spreader bar. A pipe
5 contact element 103, made of a material which does not interfere with the magnetic field created
by the magnets, is replaceably carried by the bottom face of each magnet, as shown in FIG. 23.
Each contact element has a sloping or accurately curved face which opens downwardly and also
toward the adjacent element 103. The material which defines the pipe contact elements is softer
than the metal of the pipe stands 43 to prevent the pipe stands from being scratched by the
10 contact elements. Similarly, all other components of the racker (and also of the skate and pin
carts, the low and high lift mechanisms, and the pipe stabber, e.g.) which can or will contact a
pipe stand either are made of materials which are softer than the pipe stands, or have their pipe
stand contacting surfaces defined by relatively soft and replaceable wear pieces or inserts.

Each lift head 47 preferably also includes a pair of safety latch assemblies 105; see FIG.
15 22. A latch assembly 105 can include a vertically slidable upper jaw member 106 carried by the
adjacent end of the magnet assembly. Each assembly 105 also can include a lower jaw member
107 which is pivotable about an axle 108 fixed to the magnet assembly and moved about that
axle by operation of a double acting ram 109 pinned between a crank arm of a jaw member 107
and a plate 110 carried atop the magnet assembly. The lower jaw member is moved to the side
of the bottom recessed end of the upper jaw member when the lift head is being moved into or
20 out of contact with a pipestand, but otherwise is in the position shown in FIG. 22 when the lift
head has contact with a pipe stand.

Each lift head assembly 47 can be mounted to its bridge crane spreader bar 46 by a pair
of plates 112 and 113. Plate 112 is affixed to the upper central part of the head's magnet
assembly 101 and preferably is in a plane parallel to the length of the magnet assembly. Plate
25 112 can be pinned, as at 114, to a lower tongue of plate 113 which is carried by the spreader bar.
Plate 113 can be pinned, as at 115, to the spreader bar via a vertical slot 116 in that plate. In view
of the pinned connection between plates 112 and 113, the lift head 47 can pivot about pin 114
and adjust itself to the attitude of a pipe stand being picked up either from a pipe storage bay or
from the skate cart. Also, the slidable lost-motion connection of plate 113 to the spreader bar
30 allows the connection of the lift head to the spreader bar to accommodate downward movement
of the spreader bar after the lift head has made contact with a pipe stand. Preferably a shock
absorber 118 is provided at each end of a lift head 47 to cushion movement of the spreader bar
toward the lift head. The shock absorber can be a dashpot whose piston shaft is urged upwardly
by a compression spring engaged between the upper end of the shaft and the dashpot body; see
35 FIG. 21.

Each latch assembly 105 (see FIG. 22) is a backup mechanical holder and safety
mechanism in the lift head; it is not relied upon, during normal operation of system 10, to support
a pipe stand being transported by a racker bridge crane. The several magnet assemblies carried

1 by a bridge crane are intended to hold a pipe stand in the crane. As noted above, the magnetic
forces which hold a pipe stand in a bridge crane preferably are generated by permanent magnets.
An electrically powered and selectively operable degausser 120 is included in each lift head. The
degausser is operated to null the field created by the permanent magnets when a pipe stand is to
5 be released from a bridge crane or is to be picked up by the crane.

Electrical and pneumatic power is supplied to the lift heads of each bridge crane from the
crane body. An electrical power cable 122 is connected between a reel 123 on the crane body and
the crane spreader bar 46, as shown in FIG. 17. Compressed air supply 124 and return 125 lines
are connected from reels 126 and 127 on the crane body to the spreader bar.

10 As also shown in FIG. 17, a pair of guide roller arrays 128 are carried in the top and
bottom of the truss-like body of each bridge crane in cooperation with each crane lift column 45.
There preferably are eight rollers in each array, and they engage a corresponding wear bar 129
(see FIG. 22) which extends along the length of the lift column. The lift columns preferably are
square in cross-section. There preferably is a wear bar 129 along each corner margin of each face
15 of a lift column. The guide roller arrays cooperate with the lift columns to constrain and hold the
columns stiff against pendulous motions relative to the crane body and to move essentially only
normal to the bridge crane. Such pendulous motions interfere with the ability of the crane to
pickup and deliver a pipe stand from and to a specified aligned set of sleeper notches 41 in a
desired storage bin. The presence of the wear bars on the lift columns also reinforce and stiffen
20 the lift columns themselves.

The structures and mechanisms of a presently preferred pipe storage and handling system
are constructed for operation in the temperature range of 14°F to 100°F (-10°C to 38°C). Those
structures and mechanism reflect three different sets of environmental conditions, namely;
operating conditions in which heave motions of +/- 3.7 meters at 8 second period, roll motions
25 of +/- 4° at 12 seconds, and pitch motions of +/- 4° at 9 seconds, are tolerable; waiting on weather
conditions (in which system components are stowed in their normal secure and stowed positions)
of +/- 60 meters heave at 9 seconds, +/- 10° roll at 12 seconds, and +/- 6° pitch at 10 seconds;
and survival conditions (sea fastenings in place or engaged) of +/- 8.3 meters heave at 10
seconds, +/- 35° roll at 15 seconds, and 10° pitch at 10 seconds. Also, the system components
30 are sized to withstand loads associated with drillship motions during transit of 36 feet heave, 25°
roll and 10° pitch at periods of 13-15 seconds.

The design operating rate of the system is one 5000 pound triple pipe stand per minute,
both for delivery of stands to the drill floor and removal of stands from the drill floor, using both
bridge cranes in synchronism. The bridge cranes have a nominal working speed of 60 to 75 feet
35 per minute when fully loaded, and also a low speed mode in the range of 2 to 4 from when
subjected to side forces of 0.5g opposing crane motion. When subjected to opposing side loads
in the range of 0.5 to 1.0g, the crane drives may stall but they hold the cranes in position. The
vertical drive mechanisms of the cranes have a nominal operating speed of 75 to 90 feet per

1 minute, with a slow speed mode in the range of 3 to 5 fpm useful when opposing dynamic loads
of +/- 0.5g are present due to ship motion. The crane vertical drives may stall but they hold the
vertical position when subject to opposing dynamic forces from 0.5 to 1.0g. Horizontal and
vertical drive mechanisms in the cranes have brakes which lock the crane components in position
5 when not being actively driven by an operator or by the control system ; the crane brakes are
effective under both operating and waiting on weather conditions.

The bridge cranes can be operated from either one of two control stations. A primary
control station 138 is provided in the driller's house 139 at the drilling rig floor; see FIG. 2A.
That station is called the pipe handling control console. A secondary control station 140, called
10 the pipe racker control console, is located on a platform 141 adjacent the forward end of the skate
cart truss; see FIG. 2B. During normal operation of system 10, the cranes are operated from the
primary control station in the automatic mode. The cranes are operable in either a manual mode
or the automatic mode from the secondary control station. If a crane is not selected for operation,
it is stowed in its parking position outboard of the pipe storage bays. The crane controls include
15 a zone control feature which prevents a crane from hitting structure in either operating mode.
Control interlocks prevent a crane from hitting the skate with a pipe when the skate is in motion.
When operating in the automatic mode, the crane halts above the transfer position of the skate;
a manual command is required to lower or retrieve pipe to or from the skate. When a pipe stand
is released or retrieved at the skate, the crane's automatic cycle resumes to pick up or stow the
20 next stand of pipe from or in the desired pipe storage bay.

Relevant control system interlocks are a crane interlock which prevents the cranes from
running into each other, a skate interlock which halts the crane above the skate track until a
manual command is given to the crane, a sleeper interlock which prevents a crane from lowering
its pipe lifting heads to pick up height over the sleepers until the proper sleepers are in the correct
25 position for the operation, and the structural zone interlock described above.

Controls for the sleeper indexing mechanisms preferably are located at the forward
secondary control console which is so positioned that the operator can see clearly into both pipe
bays or can move easily to see into either of them. The sleeper-related controls at that console
preferably include controls for the following functions: select port or starboard bay, select drill
30 pipe layer (1, 2, 3, ...), stowed or deployed position of the sleepers for each pipe layer,
commanding stow or deploy sleeper motion, an alarm (sound and light) which indicates that one
or more sleepers did not respond to a positioning command, and a sleeper lock useful when
personnel are present or racked pipe stands. The sleeper stow and position commands signals
preferably are also provided to the crane control programming pertinent to the sleeper control
35 interlock.

The skate cart and pin cart drive rates are defined consistent with the one per minute pipe
stand delivery/recovery rate. The skate drive preferably is operable to produce skate speeds up
to 450 feet per minute and to accelerate from a dead stop to full speed in 4 seconds. The pin cart

drive preferably is operable to produce pin cart speeds up to 600 feet per minute, and to accelerate the pin cart when unloaded from a dead stop to full speed in 2 seconds. The pin cart drive forces the pin cart into contact with the pin end of a pipe stand while the stand is being raised by the derrick elevators without pushing the stand through the elevators. As noted above, an overrunning clutch in the pin cart drive enables the pin cart to overrun its drive in response to loads on the cart by a pipe stand being placed on the skate cart. The skate cart and pin cart drives preferably are of the variable speed kind. The skate cart can make a 112 foot round trip in one minute on a continuous basis. The pin cart travel is about 80 feet along the skate cart. Its drive can return the empty pin cart to its receiving position in 30 seconds. A brake for the skate cart can stop the skate cart in two seconds once per minute when loaded and twice per minute when unloaded. A pin cart brake can halt the pin cart from full speed in one second, and can effect two braking operations per minute on a continuous basis. There are shock absorbers at the limits of travel of the skate and pin carts. The skate and pin cart drives are controllable from primary control station 138.

Controls for operating pipe low lift mechanism 52, pipe stabber 73 and pipe high lift mechanism 80 are located at primary control station 138. The high lift mechanism can lift the box of a pipe stand to an elevation of 22 feet above the rotary table located in the drilling floor at the drilling axis. The low and high lift mechanism can lift 2500 pounds. The low lift operating rate is 4 feet per second. The high lift operating rate is 2 feet per second.

Further information about the control functions and equipment of the presently preferred system 10 are set forth in Tables I and II, pertinent to control stations 138 and 140, respectively.

The information set forth in the foregoing description and in the accompanying Figures and Tables is not exhaustive of all forms which structures and procedures according to this invention may take. Variations and modifications of the described structures and procedures will occur to persons skilled in the relevant arts and technologies, and they are within the fair scope and meaning of the following claims.

Table I:**Control Functions and Operator Interfaces for the Pipe Handling Control****Console**

Control Function	Operator interface	Action/Response	Priority
Skate Drive	Proportional Joystick	Left-Skate moves left Center-Skate Stops Right-Skate moves right	Primary Control (Duplicate)
Skate Emergency Stop	Push-pull button	Push-Set Skate brake Pull-Release skate brake	Primary Control Shared (Duplicate)
Pipe Lifter	Push and hold buttons	Push "UP"-raise lifter Release "UP"-stop Push "DOWN"-Lower lifter Release "DOWN"-stop	Primary control (Singular)
Pipe Stabber	Proportional Joystick and rocker switch control buttons	Left-Stabber Extends Center-stabber stops Right Stabber retracts Depress Lt. Button "O"- Jaws open Depress Button "C"- Jaws close	Primary control (Singular)
High Lift	Push and hold buttons	Push "UP" - Lifter raises Push "DOWN"-Lifter lowers	Primary control (Singular)
Port Bridge Crane pipe pickup & release	Push and hold Buttons and rocker switch	Push "UP" to raise crane Release "UP" to stop Push "DOWN" to lower crane Release "DOWN" to stop Rock to "P" - Pick pipe Rock to "R" - Release pipe	Primary control (Duplicate)

Stbd Crane pipe pickup & release	Push and hold buttons and rocker switch	Push "UP" to raise crane Release "UP" to stop Push "DOWN" to lower crane Release "DOWN" to stop Rock to "P" - Pick pipe Rock to "R" - to release pipe	Primary control (Duplicate)
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Table II**Control Functions and Operator Interface for the Pipe Racker Control****Console**

<u>Control Function</u>	<u>Operator Interface</u>	<u>Action/Response</u>	<u>Priority</u>
Skate Drive	Proportional Joystick	Left-Skate moves left Center-Skate Stops Right-Skate moves right	Backup Control Maintenance (Duplicate)
Skate Emergency Stop	Push-pull button	Push-Set Skate brake Pull-Release skate brake	Primary Control Shared (Duplicate)
Port Bridge Crane pipe pickup & release at skate	Push and hold buttons and rocker switch	Push "UP" to raise crane Release "UP" to stop Push "DOWN" to lower crane Release "DOWN" to stop Rock to "P" - Pick pipe Rock to "R" - to release pipe	Backup control Pipe Transfer Maintenance (Duplicate)
Stbd Bridge Crane pipe pickup/release at skate	Push and hold buttons and rocker switch	Push "UP" to raise crane Release "UP" to stop Push "DOWN" to lower crane Release "DOWN" to stop Rock to "P" - Pick pipe Rock to "R" - to release pipe	Backup control Pipe Transfer Maintenance (Duplicate)

Port Bridge Crane side to side and vertical movement	Proportional Joystick (Two speed push and old button optional)	Center-crane stops Push port-Move port Push stbd-move stbd Push away - lower Pull to - raise	Backup manual control (Duplicates automatic control)
Stbd Bridge Crane side to side and vertical movement	Proportional Joystick (Two speed push and hold button in place of joy stick optional)	Center-crane stops Push port-Move port Push stbd-move stbd Push away - lower Pull to - raise	Backup manual control (Duplicates automatic control)
Port Bridge Crane pipe pick-up and release in racker	Two position rocker switch	Rock to "P" - pick pipe Rock to "R" - release pipe	Backup manual control (Duplicates automatic control)
Stbd Bridge Crane pipe pick-up and release in racker	Two position rocker switch	Rock to "P" - pick pipe Rock to "R" - release pipe	Manual Operation Primary Control (singular)
Port Bay Sleeper Level Selection	18 (17) count selector switch	Position to sleeper level	Backup manual control (Duplicates automatic control)
Stbd Bay Sleeper Level Selection	18 count selector switch (May be incorporated with Set/Stow switches)	Position to sleeper level	Backup manual control (Duplicates automatic control)
Port Sleeper Set/Stow	Rocker Switch	Rock to "Stow" - Sleepers stow Rock to "Set" - Sleepers set	Backup manual control (Duplicates automatic control)

Stbd Sleeper Set/Stow	Rocker Switch	Rock to "Stow" - Sleepers stow Rock to "Set" - Sleepers set	Backup manual control (Duplicates automatic control)
Mode Selection	Rocker Switch	Rock to "Automatic" - Automatic control Rock to "Manual" - Manual control	Primary Control (Singular)
Automatic Control	PLC control	Automatically operates port and stbd cranes from racker to transfer position	Primary Control (Duplicates all manual operations which duplicate automatic control functions)

1 CLAIMS

1. A drill pipe storage apparatus comprising:
a pipe storage bin,
horizontal pipe support members disposable in the bin at plural spaced stations
along the length of the bin for individually supporting plural horizontal lengths of drill pipe in
an array of plural vertically spaced layers and of plural lengths of drill pipe in each layer, and
selectively operable drive mechanisms connected to the pipe support members and
operable to move the members individually between deployed positions in which the support
members are in the array and horizontal retracted positions in which the support members are
removed from the array.

2. Apparatus according to claim 1 in which the pipe support members are arranged
to support the pipe lengths without subjecting any pipe length to loads due to pipe and support
members thereabove in the array.

3. Apparatus according to claim 1 in which the lowermost support member at each
station as carried on a bin base, and each other support member at the station when disposed in
the array is engaged with and supported on the support member below it.

4. Apparatus according to claim 3 in which the pipe support members when engaged
with each other are keyed against relative movement in directions along pipe lengths supported
in the bin.

5. A drill pipe storage apparatus comprising:
a pipe storage bin,
horizontal pipe support members disposable in the bin at plural spaced stations
along the length of the bin for individually supporting plural horizontal lengths of drill pipe in
an array of plural vertically spaced layers and of plural lengths of drill pipe in each layer, the
lowermost support member at each station being carried on a bin base, and each other support
member at the station when disposed in the array being engaged with and supported on the
support member below it, each pipe support member defining in an upper part thereof a plurality
of upwardly open notches sized in cooperation with the vertically adjacent contour of the support
member thereabove in the array to receive in each notch a respective pipe length of selected
diameter without contact of the pipe length with the support member directly thereabove, and
selectively operable drive mechanisms connected to the pipe support members and
operable to move the members individually between deployed positions in which the support

1 members are in the array and retracted positions in which the support members are removed from
the array.

5 6. Apparatus according to claim 5 in which the pipe receiving notches have
substantially straight sloping sides.

7. A drill pipe storage apparatus comprising:
a pipe storage bin,
horizontal pipe support members disposable in the bin at plural spaced stations
10 along the length of the bin for individually supporting plural horizontal lengths of drill pipe in
an array of plural vertically spaced layers and of plural lengths of drill pipe in each layer, and
selectively operable drive mechanisms connected to the pipe support members and
operable to turn the support members about vertical axes located outside the array to move the
members individually between deployed positions in which the support members are in the array
15 and retracted positions in which the support members are removed from the array.

8. Apparatus according to claim 7 in which the drive mechanisms are operable to lift
and lower the pipe support members.

20 9. Apparatus according to claim 7 in which the drive mechanisms include at each
station a rotatable vertical shaft with which is associated a group of pipe support members, each
support member in the group having an end frame through which the shaft rotatably passes, a
coupling selectively engageable between each pipe support member and the shaft for securing
the shaft from rotation relative to the pipe support member, and a shaft drive operable for rotating
25 the shaft a selected amount in either direction about its axis.

30 10. Apparatus according to claim 9 in which the shaft is axially movable through each
pipe support member, each coupling is operable for securing the associated pipe support member
from axial motion of the shaft relative thereto, and the shaft drive is operable for raising and
lowering the shaft a selected amount.

11. Apparatus according to claim 9 including a holder for each support member with
which the support member is engageable in its retracted position.

35 12. Apparatus according to claim 9 in which the pipe support members at each station
comprises two groups of movable support members, alternate support members being members
of a respective group with which is associated a respective one of a pair of vertical shafts.

1 13. Apparatus according to claim 12 in which the shafts at each station are disposed at a common side of the bin.

5 14. Apparatus according to claim 12 in which the pipe support members in one group have retracted positions in which they extend in one direction from the station substantially parallel to the array, and retracted pipe support members in the other group extend in an opposite direction from the station substantially parallel to the array.

10 15. Apparatus according to claim 12 in which the deployed positions of the pipe support members at each station are in a common vertical plane disposed transversely of the array, and each support member above the lowermost one is supportively engaged with the support members below it.

15 16. A drill pipe storage apparatus comprising:
a pipe storage bin stationary relative to a place of pipe use
horizontal pipe support members disposable in the bin at plural spaced stations along the length of the bin for individually supporting plural horizontal lengths of drill pipe in an array of plural vertically spaced layers and of plural lengths of drill pipe in each layer,
20 selectively operable drive mechanisms connected to the pipe support members and operable to move the members individually between deployed positions in which the support members are in the array and retracted positions in which the support members are removed from the array, and a pipe lifter disposable above the array and operable to move individual pipe lengths in a horizontal attitude between the array and a transfer position laterally of the array.

25 17. Apparatus according to claim 16 in which the pipe lifter comprises a bridge crane spanning the length of the bin and movable transversely relative to the bin.

30 18. Apparatus according to claim 16 in which the pipe lifter includes a plurality of controllable magnetic pipe lift units engageable with a pipe length at spaced locations along the length.

19. Apparatus according to claim 18 in which each pipe lift unit comprises plural permanent magnets and a selectively operable degausser.

35 20. Apparatus according to claim 18 in which each pipe lift unit includes a backup mechanical holder selectively engageable with and releasable from a pipe length.

21. Apparatus according to claim 18 in which the pipe lift units are supported on a common carrier.

22. Apparatus according to claim 21 in which the bridge crane has a stowed position spaced laterally from the bin in which pipe lift units are securable to structures at a lower exterior portion of the bin.

23. Apparatus according to claim 21 in which the common carrier for the pipe lift units is movable vertically relative to the bridge crane via a plurality of column members drivable vertically of the crane.

24. Apparatus according to claim 23 in which the column members have vertically spaced guides in the bridge crane arranged to constrain the column members to motion substantially only normal relative to the crane.

25. Apparatus according to claim 16 in which the bin is spaced in a direction substantially parallel to the length of the bin from a place of use of pipe lengths, and including a pipe delivery mechanism for moving pipe in a horizontal attitude between a transfer position adjacent the bin and the place of pipe use, the pipe delivery mechanism includes a track extending from the transfer position toward the place of pipe use.

26. Apparatus according to claim 25 including an elongate carriage drivable in each of two opposite directions along the track to and from the transfer position, the carriage having a length adequate to support a pipe length in alignment therewith at one end of the pipe length and at a location along the pipe length near its other end.

27. Apparatus according to claim 26 including a cart drivable in each of two opposite directions along the length of the carriage, the cart defining an upwardly open receptacle for receiving and bearing the one end of a pipe length supported on the carriage.

28. Apparatus according to claim 27 including a pipe support roller mounted at the end of the carriage nearest the place of pipe use for rotation about a horizontal axis.

29. Apparatus according to claim 28 in which the roller has a larger diameter in its ends than between its ends.

1 30. Apparatus according to claim 28 including a selectively operable lift mechanism mounted between the roller and the carriage operable for controllably raising and lowering the roller relative to the carriage.

5 31. Apparatus according to claim 30 in which the roller lift mechanism is disposed on the carriage out of the path of the cart along the carriage.

32. Apparatus according to any one of claims 25-31 in which the place of pipe use is a well drilling facility which includes a drilling operations platform.

10 33. Apparatus according to claim 32 in which the track is substantially coplanar with the drilling operations platform.

15 34. Apparatus according to claim 32 in which the drilling facility is located on a floatable offshore drilling structure.

35. Apparatus according to claim 34 including a second pipe storage bin disposed in proximate parallel relation to the transfer position.

20 36. A drill pipe storage and handling apparatus for a well drilling rig comprising:
a track extending from one end adjacent the drilling rig to an opposite end remote from the rig,

25 an elongate carriage adapted to travel along the track and to receive a length of drill pipe disposed longitudinally with respect to the track and to support a received pipe length at spaced locations therealong,

30 a pipe storage bin disposed laterally of one end of the track including horizontal pipe support members cooperatively configured for individually supporting plural lengths of drill pipe in an array of plural vertically spaced layers of pipe and plural length of pipe in each layer, the pipe support members above the bottom layer being indexable between deployed positions in and transversely of the array and retracted positions outside the array,

a moveable pipe lifter disposable above the bin operable to move individual pipe lengths between the array and the carriage.

35 37. Apparatus according to claim 36 in which the carriage includes a pipe lifter at its end adjacent the drilling rig operable to lift the adjacent end of a received pipe length a selected distance above the carriage.

38. Apparatus according to claim 37 in which each pipe length has a pin end and a box end, the pipe lengths are disposed in the array with their pin ends remote from the drilling rig, and including a cart movable along the carriage adapted for supporting the pin end of a received pipe length.

39. Apparatus according to claim 38 in which the carriage is drivable along the track, and the cart is drivable along the carriage.

40. Apparatus according to claim 36 in which the track and the carriage are common to and are disposed between a pair of similar bins.

41. Apparatus according to claim 40 in which the pipe lifter is operable to move pipe lengths between either bin and the carriage.

42. A method of storing oil and gas well drill pipe comprising the steps of horizontally disposing a selected number of pipe lengths, as a first bottom layer thereof, individually in upwardly open notches in the upper extents of a set of stationary pipe supports disposed transversely of the pipe lengths at stations spaced along the lengths, and horizontally disposing further numbers of pipe lengths in further similarly notched pipe support sets placed at each station atop the supports therebelow to create a stationary array of plural layers of plural numbers of pipe lengths, and raising and lowering individual pipe lengths directly from above and to receiving notches in the pipe supports.

43. The method according to claim 42 including defining the support members so that each pipe length in the array makes contact only with the surfaces of the upwardly open notches of the pipe supports immediately below it in the array.

44. The method according to claim 42 including the further step of moving each set of pipe supports to retracted horizontal positions out of the array upon removal of all pipe lengths from the layer supported by that set to expose the next lower layer in the array, and moving the next upper set of supports into deployed positions in the array on filling a pipe length layer in the array.

45. The method according to claim 44 in which moving the pipe supports from deployed positions to retracted positions includes raising the deployed supports out of contact with the supports therebelow in the array, swinging each raised support horizontally about an axis at an end of the support, and lowering the raised and swung supports into holders therefor located outside the array.

46. The method according to claim 45 in which moving the pipe supports from retracted positions to deployed positions includes performing the reverse of each of the operations described in claim 45 in reverse sequence.

47. The method according to claim 42 in which raising individual pipe lengths from the pipe supports includes engaging a pipe length from above at spaced locations along the length by a plurality of magnetic lift heads, and raising the lift heads in substantial unison.

48. The method according to claim 42 in which lowering individual pipe lengths to the pipe supports includes horizontally supporting a pipe length from above via a plurality of magnetic lift heads at spaced locations along the pipe length, lowering the lift heads in substantial unison to place the pipe length in aligned notches in a set of pipe supports, and nulling the magnetic fields of the lift heads.

49. A method of storing oil and gas well drill pipe comprising the steps of horizontally disposing a selected number of pipe lengths, as a first bottom layer thereof, individually in upwardly open notches in the upper extents of a set of pipe supports disposed transversely of the pipe lengths at stations spaced along the lengths, and horizontally disposing further numbers of pipe lengths in further similarly notched pipe support sets placed at each station atop the supports therebelow to create an array of plural layers of plural numbers of pipe lengths, and raising and lowering individual pipe lengths directly from and to receiving notches in the pipe supports, raising a pipe length from its pipe supports including moving the pipe length in a horizontal attitude from the array to a state of support on a carriage movable along a path laterally from, adjacent to and parallel to the array, the carriage supporting the pipe length at spaced locations along the pipe length.

50. The method according to claim 49 including raising an end one of the locations of carriage support of the pipe relative to the carriage upon movement of the carriage to a selected place displaced from the array, the selected place being associated with removal of the pipe length from the carriage.

51. The method according to claim 49 in which the carriage has two locations of support of a pipe length disposed thereon, one of which is raisable relative to the carriage, the other of which is movable along the carriage and is adapted to support an end of the pipe length.

52. A method for storing, handling, and moving drill pipe in association with a well drilling rig having a drilling operations floor, the method comprising the operations of:

Art. 34

- 1 lifting a stand of drill pipe directly from an individual horizontal storage position
in an array of stand storage positions,
placing the lifted stand on a carriage arranged to support the placed stand at spaced
locations along its length,
5 moving the carriage towards the floor to place one end of the carriage at the floor,
elevating the one end of the placed stand above its placed position on the carriage
as the carriage nears the floor, and
hoisting the stand via the one end thereof to a vertical position above the floor while
movably supporting the other end of the stand on the carriage.

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53. The method according to claim 52 in which the elevating operation includes raising
the location of carriage support of the placed stand which is nearest the one end of the stand.

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International Bureau



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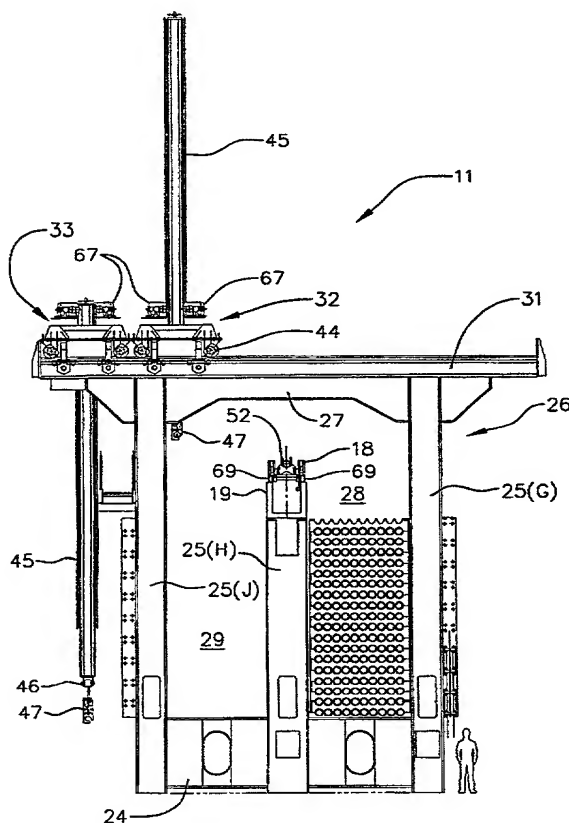
(74) Agent: **CARNEY, Hayden, A.;** Christie, Parker & Hale, LLP, P.O. Box 7068, Pasadena, CA 91109-7068 (US).

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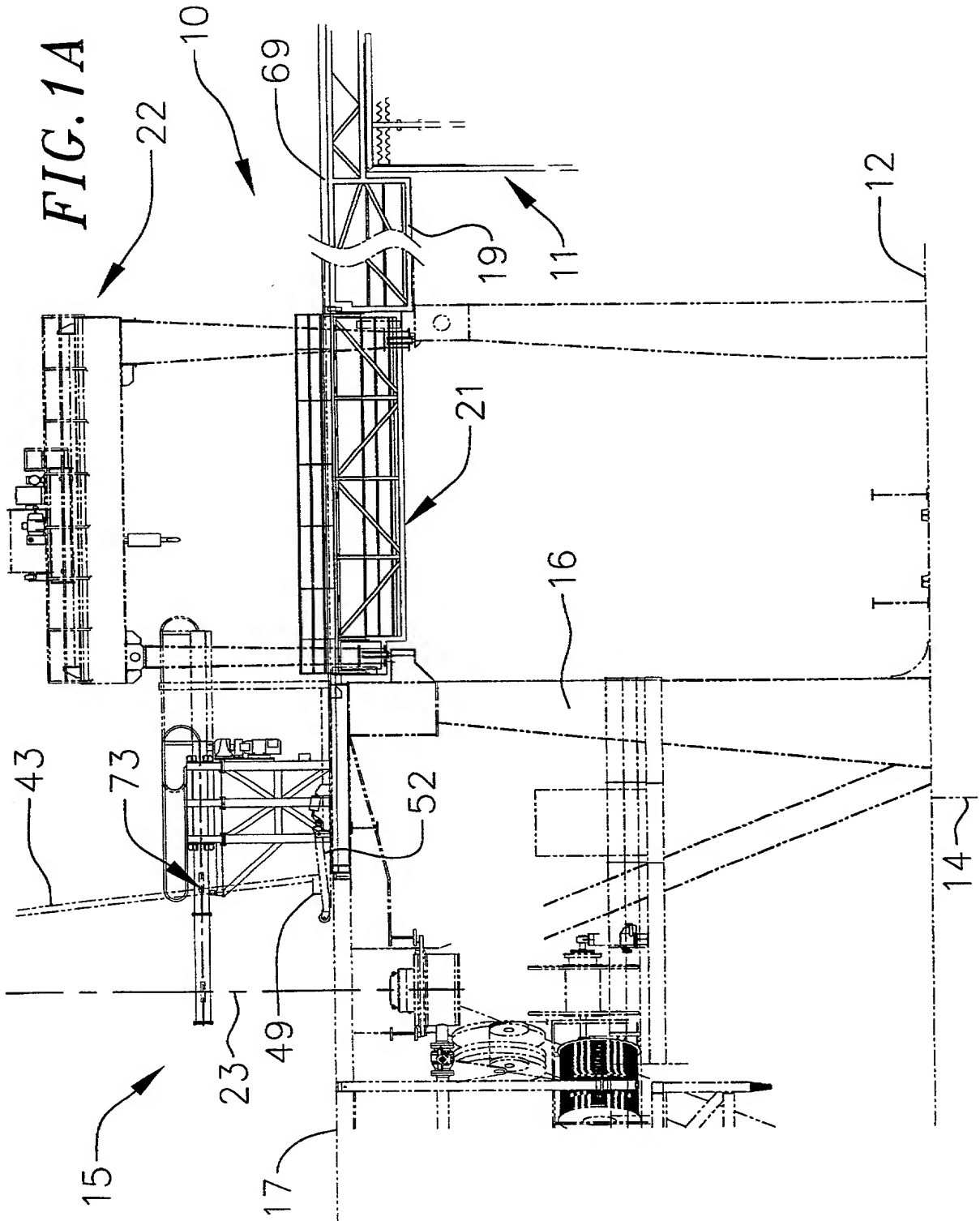
[Continued on next page]

(54) Title: HORIZONTAL DRILL PIPE RACKER AND DELIVERY SYSTEM



(57) Abstract: Stands (43) of multiple lengths of drill pipe, for oil and gas drilling are individually supported in a storage bin (28, 29) in an array of plural layers of plural numbers of stands. The individual stands are held in cooperating notches (41, 42) in vertically aligned support sleepers (38, 39) so that each stand experiences loads due only to its own weight and to motions of a bin frame (13) where, as preferred, the frame is supported on a drillship deck (12). Sleepers between stand layers are indexable between positions in the array and outside the array. A bridge crane (32, 33) spans the length of the bin and is moveable transversely above it. The bridge crane carries vertically movable magnetic lifting heads (47) for lifting, holding and releasing an individual pipe stand. The crane can move a stand between a supported position in the bin array and a supported position on a power-driven carriage (18, 49) which is movable along a path along an upper part of a side of the bin (28, 29). The carriage includes a skate cart (18) for transporting a stand as such, and a pin cart (49) drivable along the skate cart for supporting the pin end of a stand on the skate. The carriage path preferably ends inside the margin of the floor (17) of a drilling rig.

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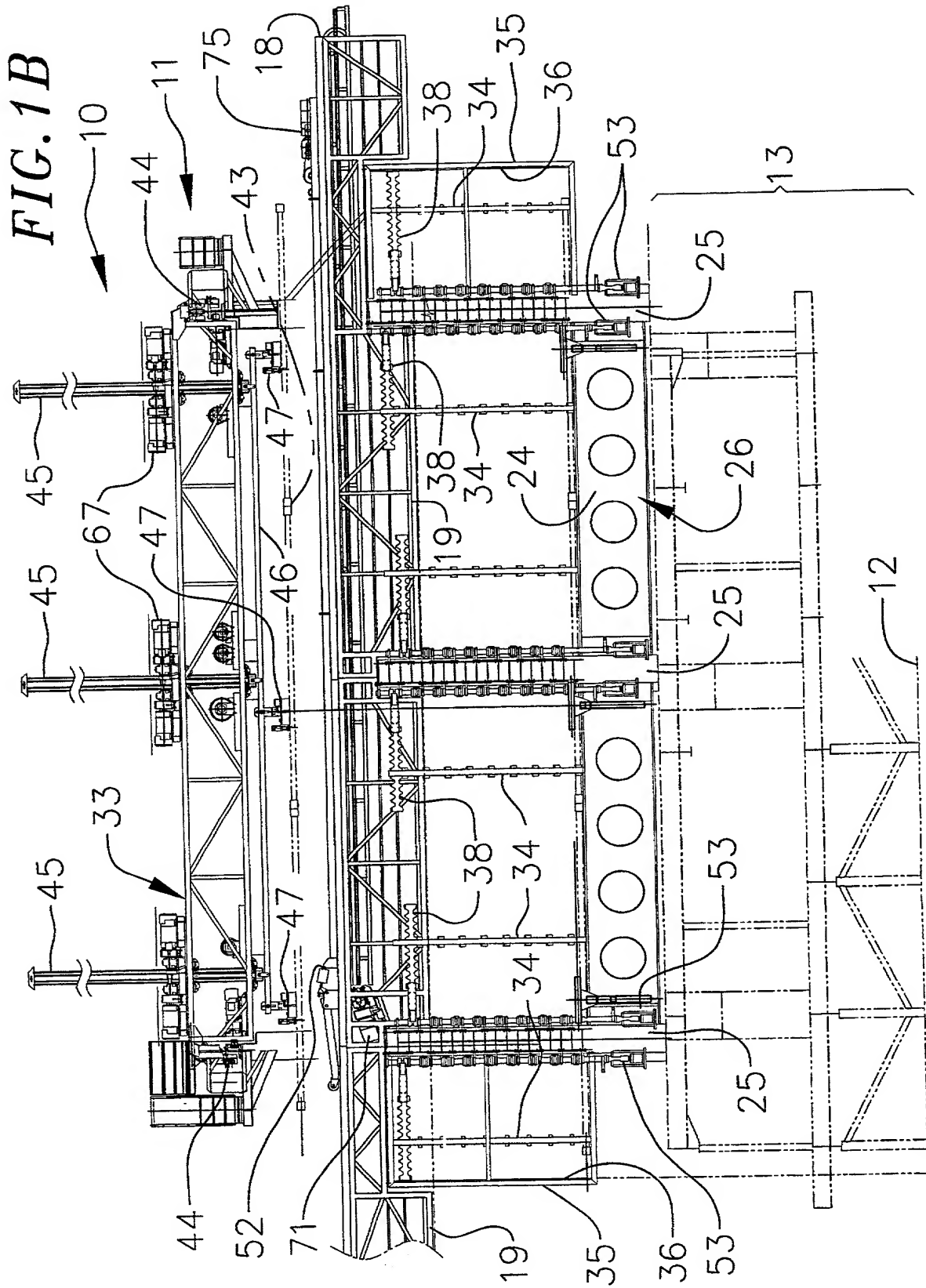
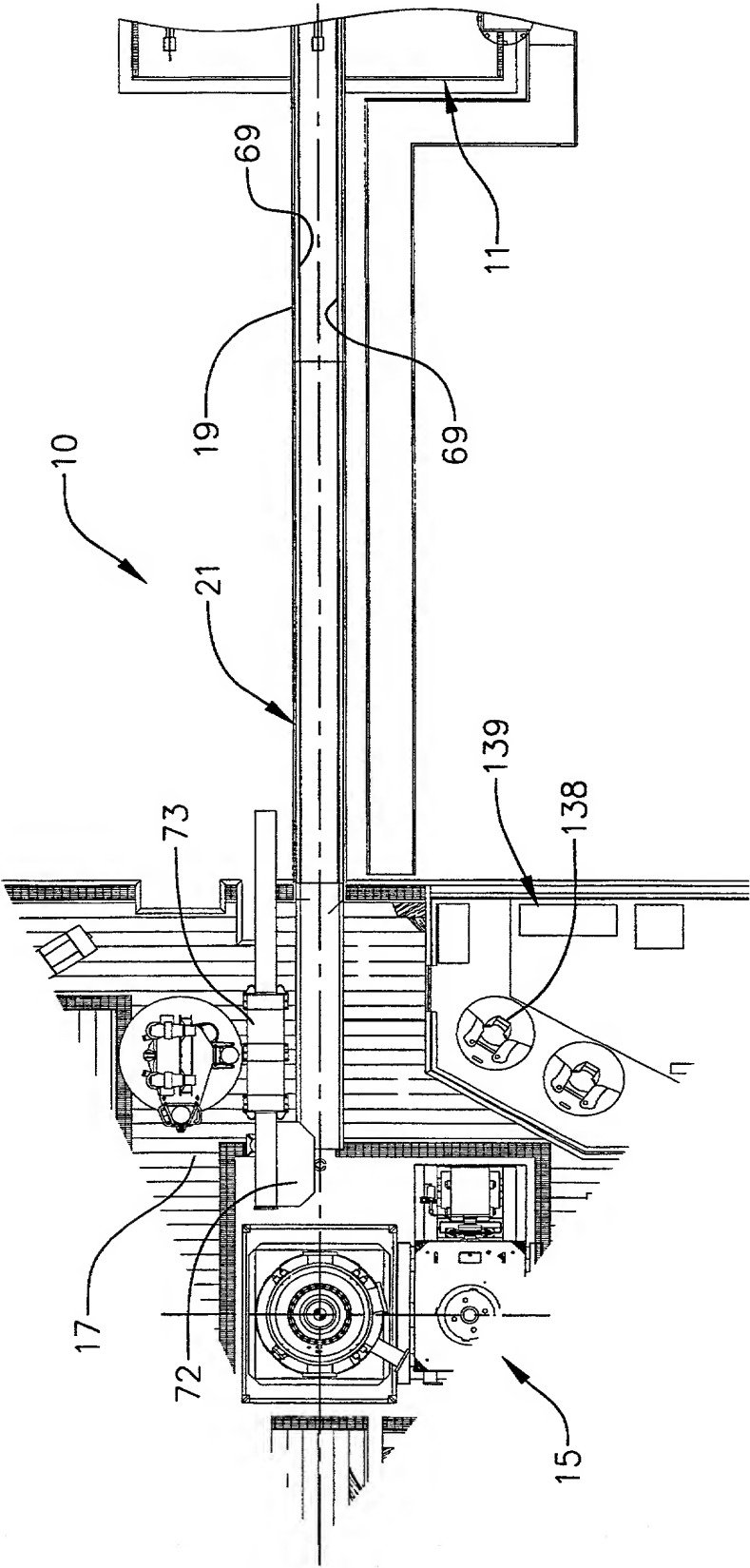


FIG. 2A



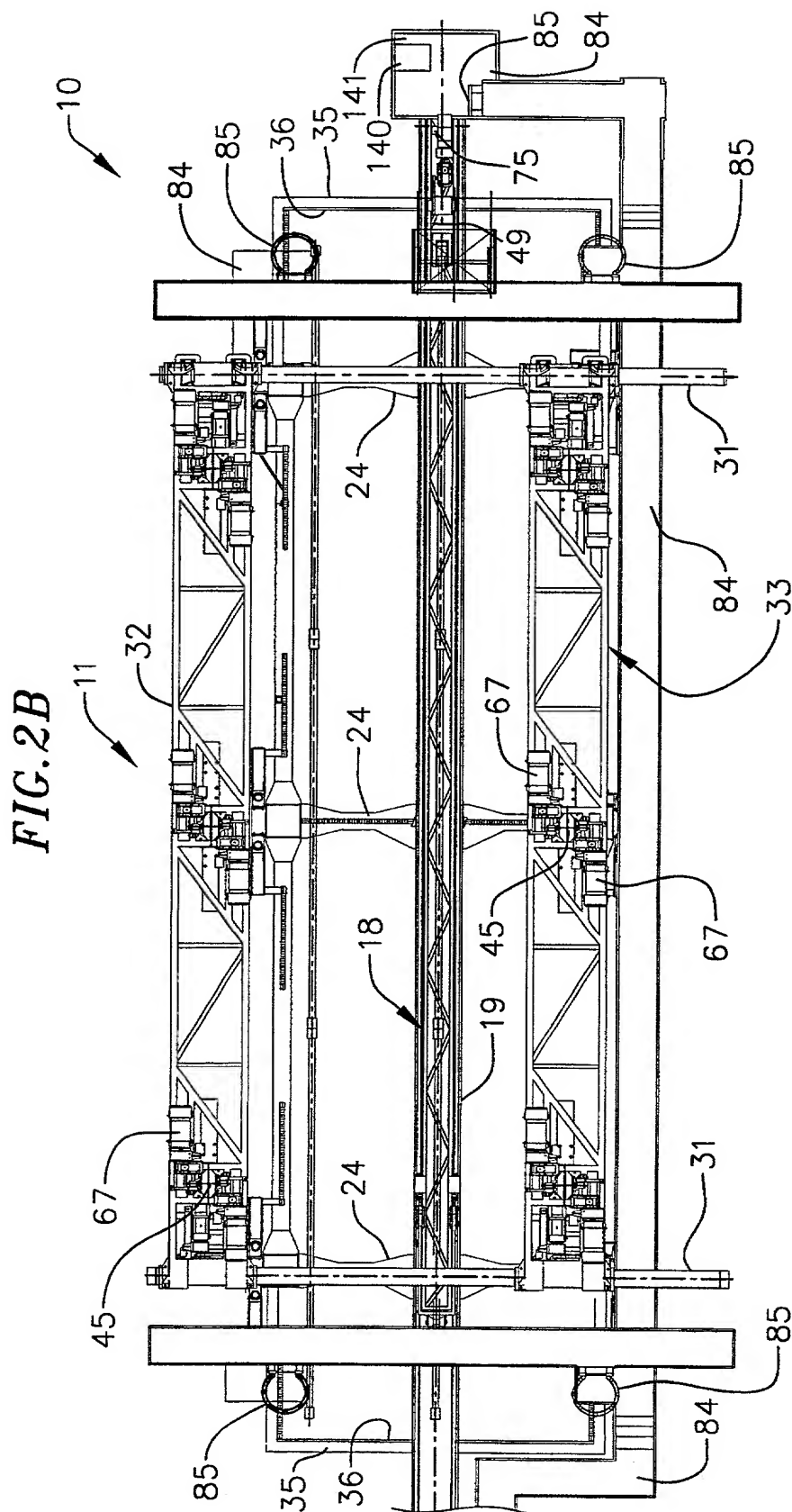


FIG.3

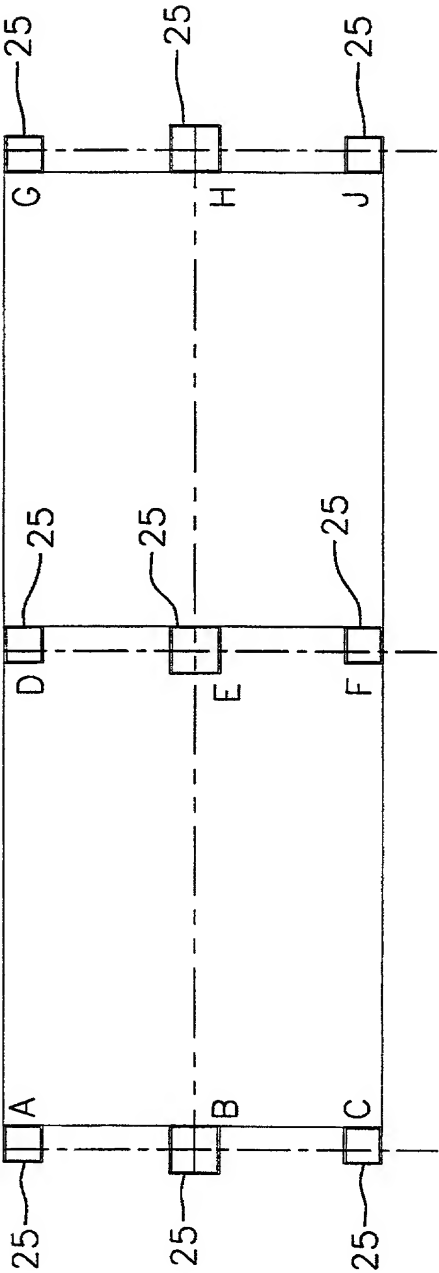


FIG. 4

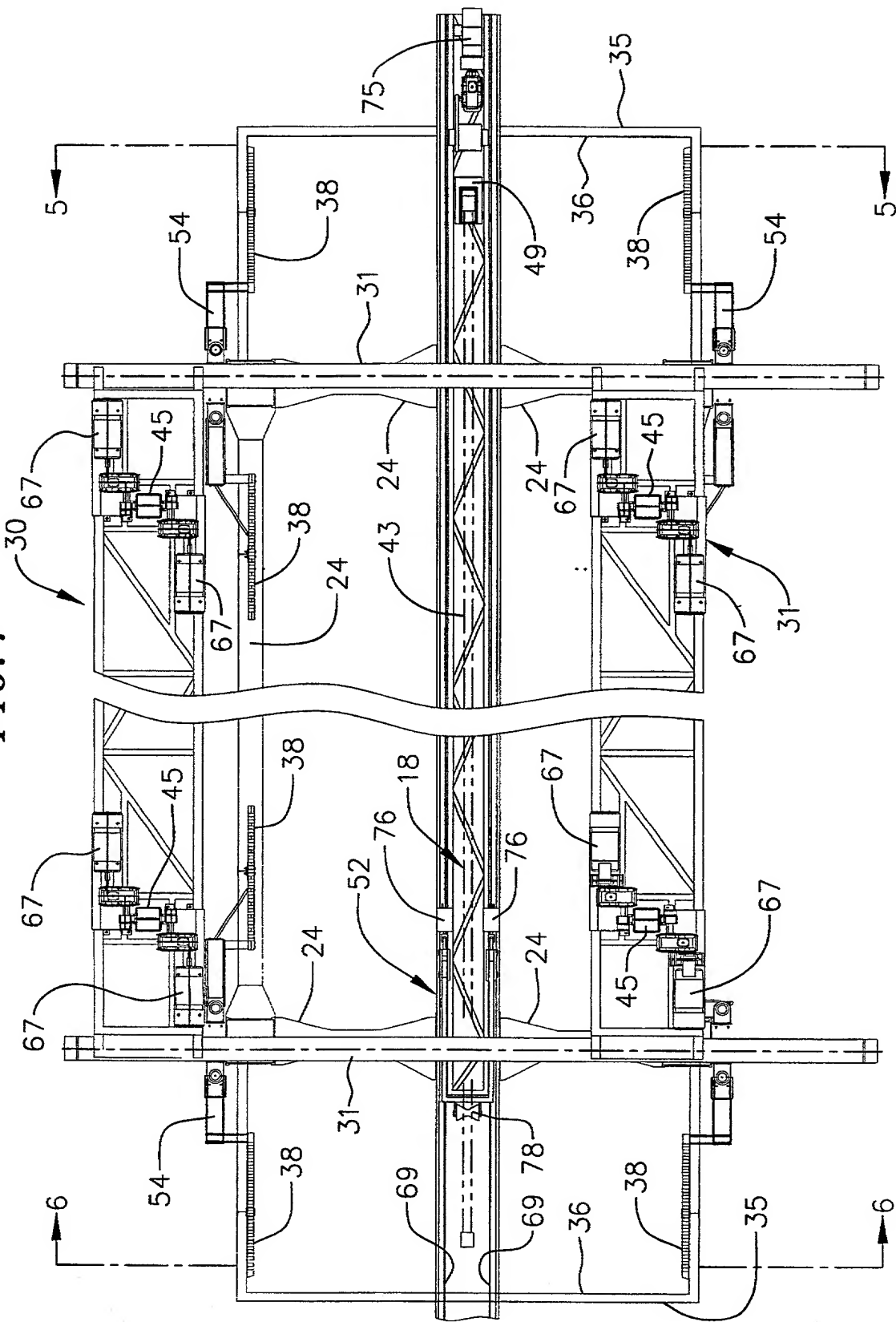


FIG.5

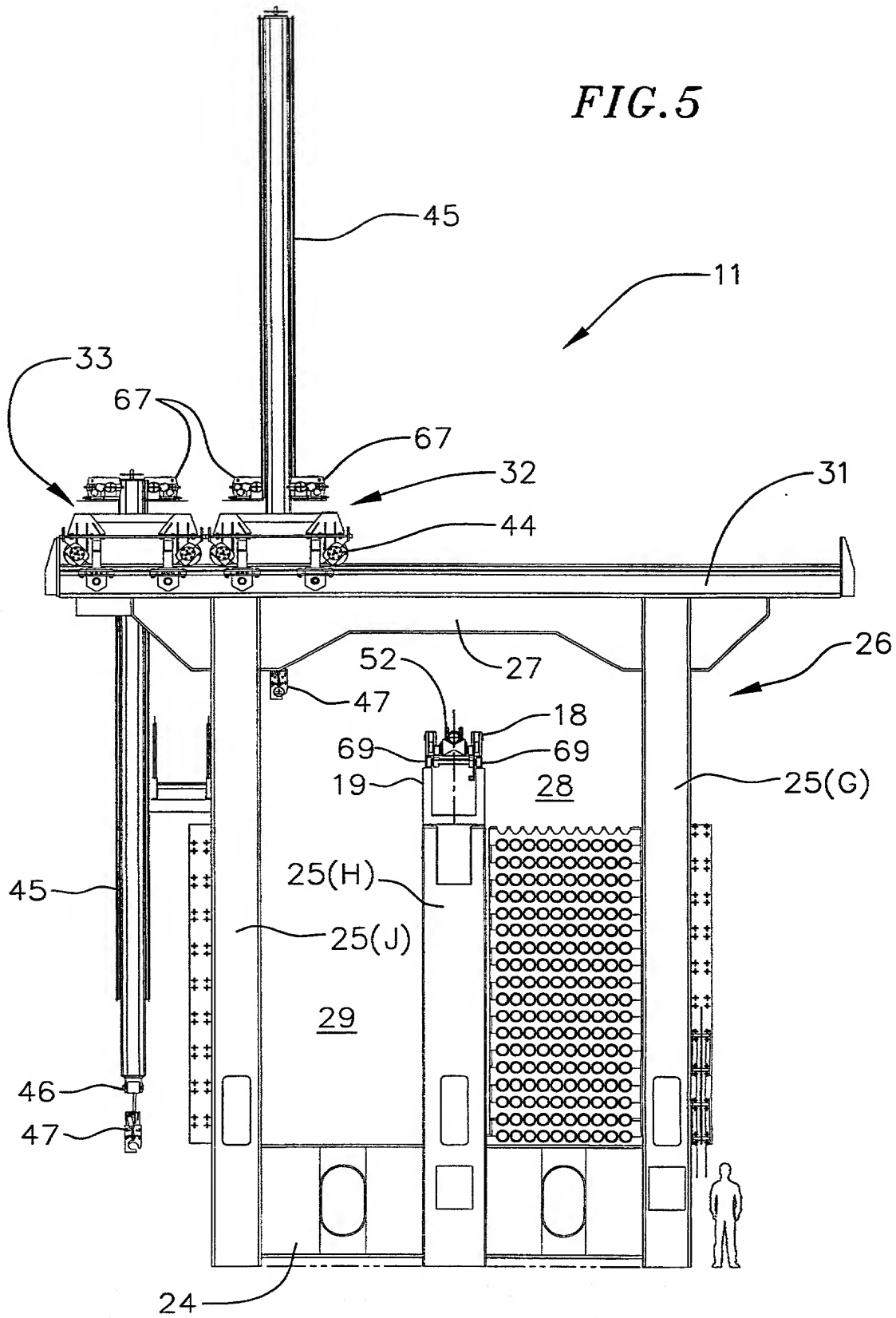


FIG. 6

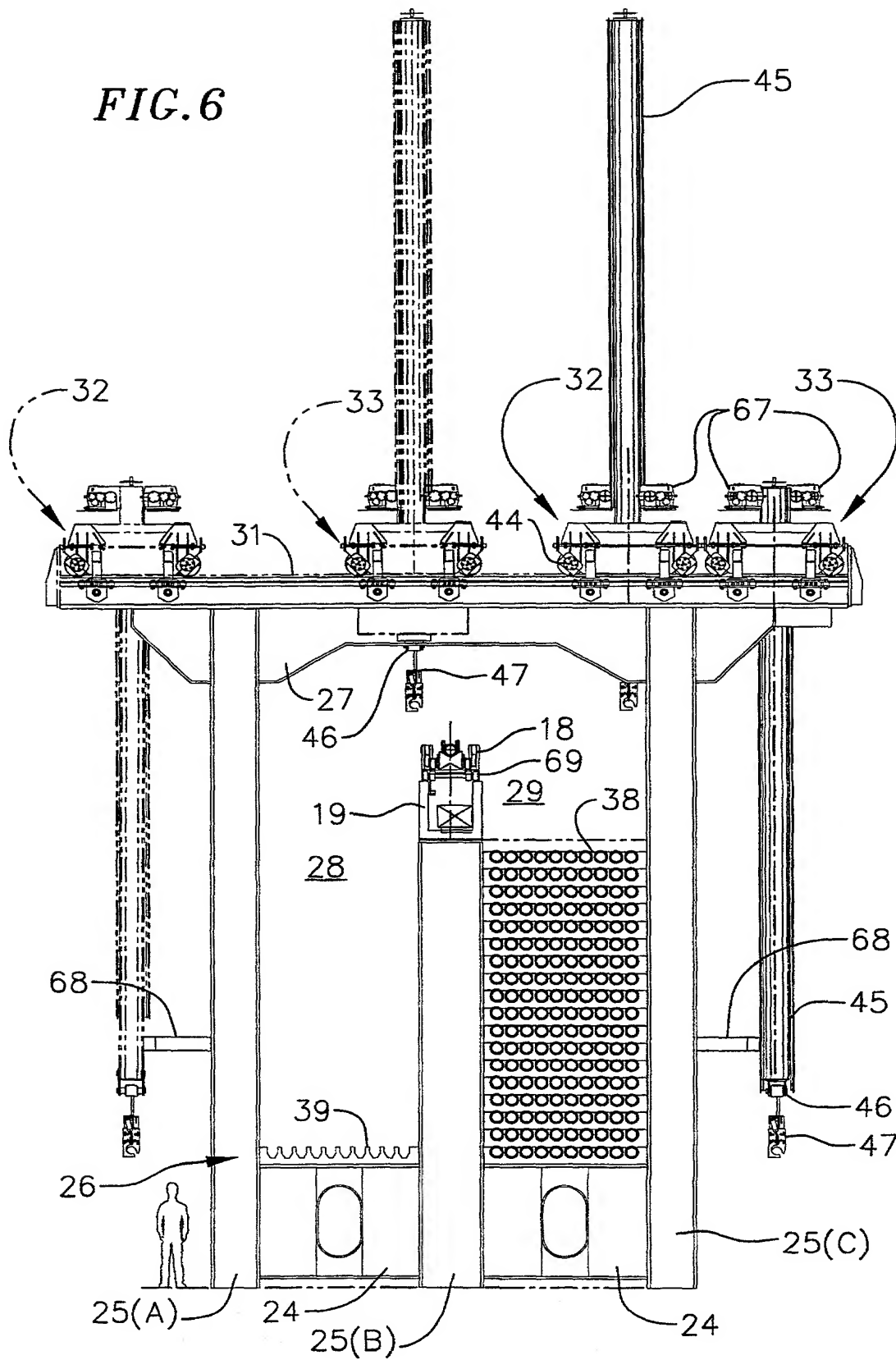


FIG. 7

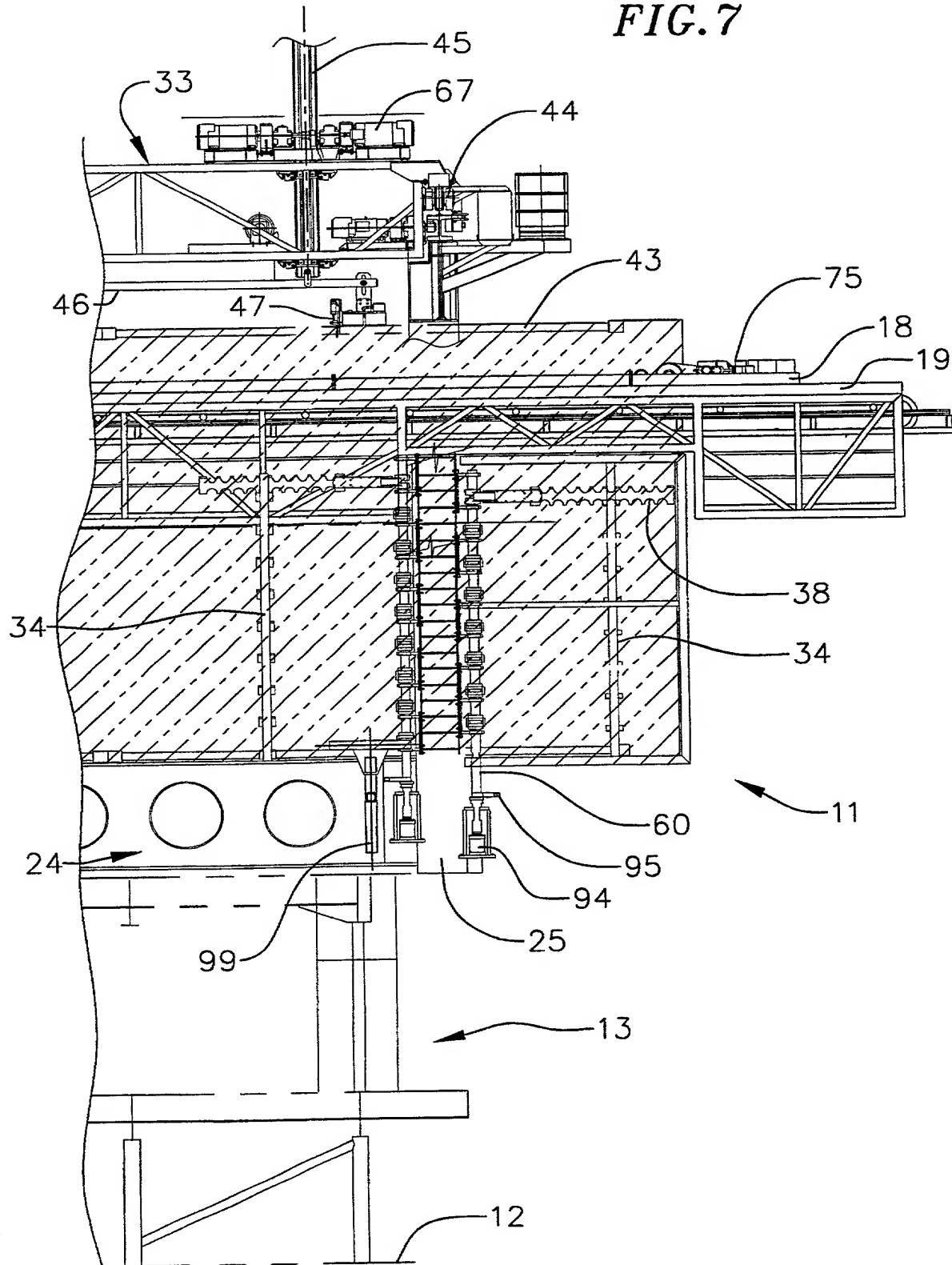


FIG. 8

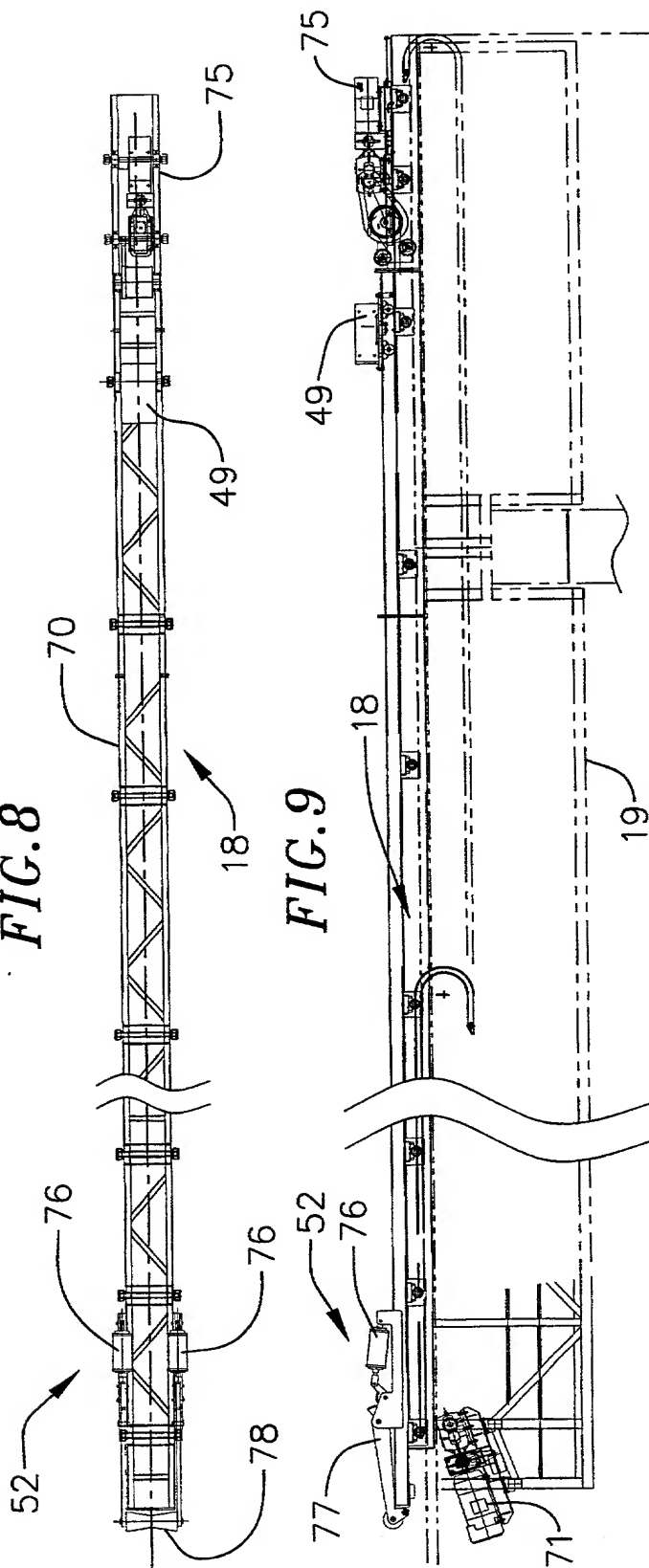


FIG. 9

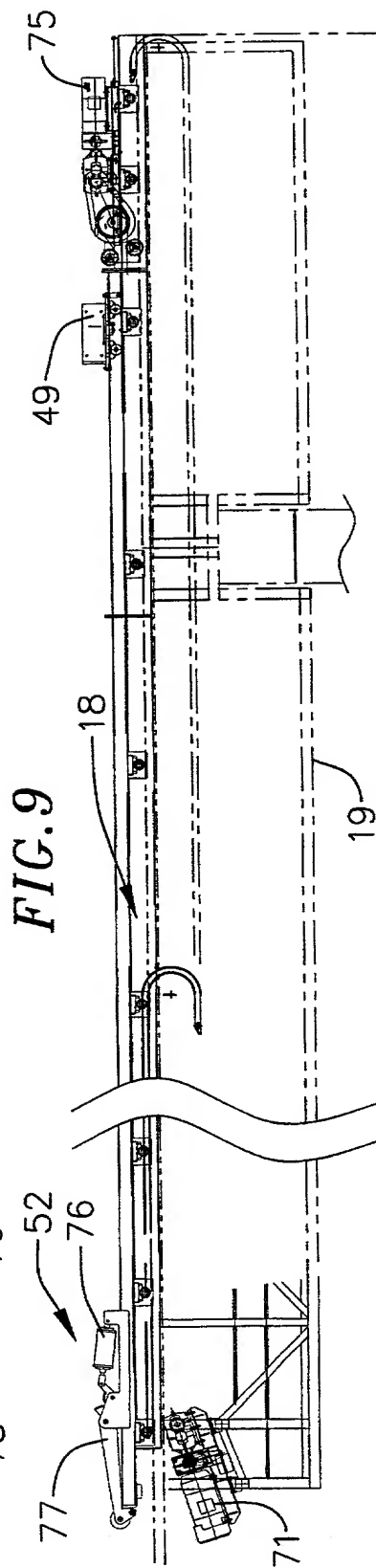
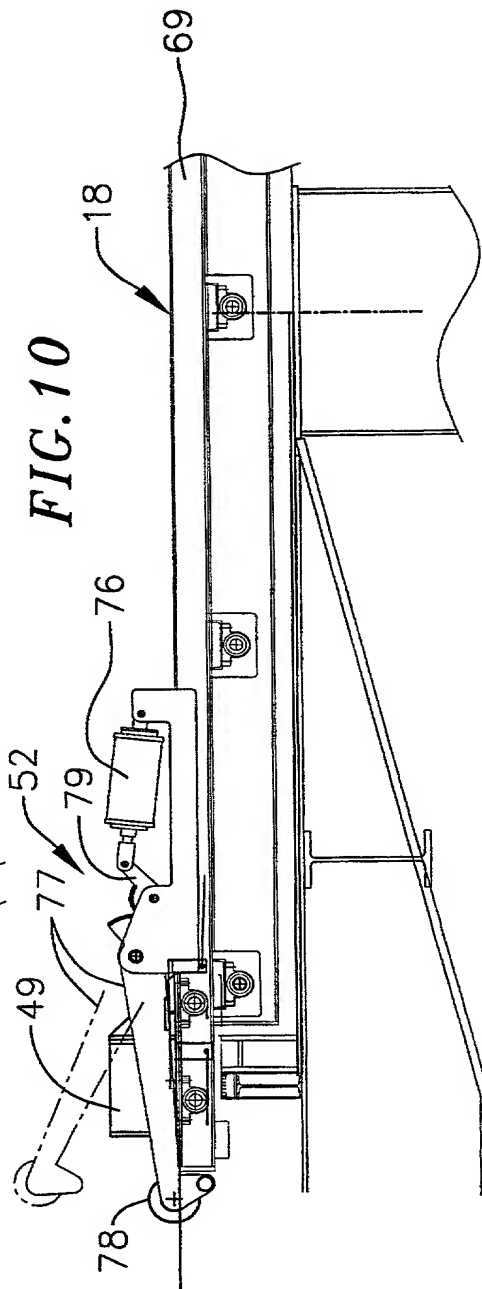


FIG. 10



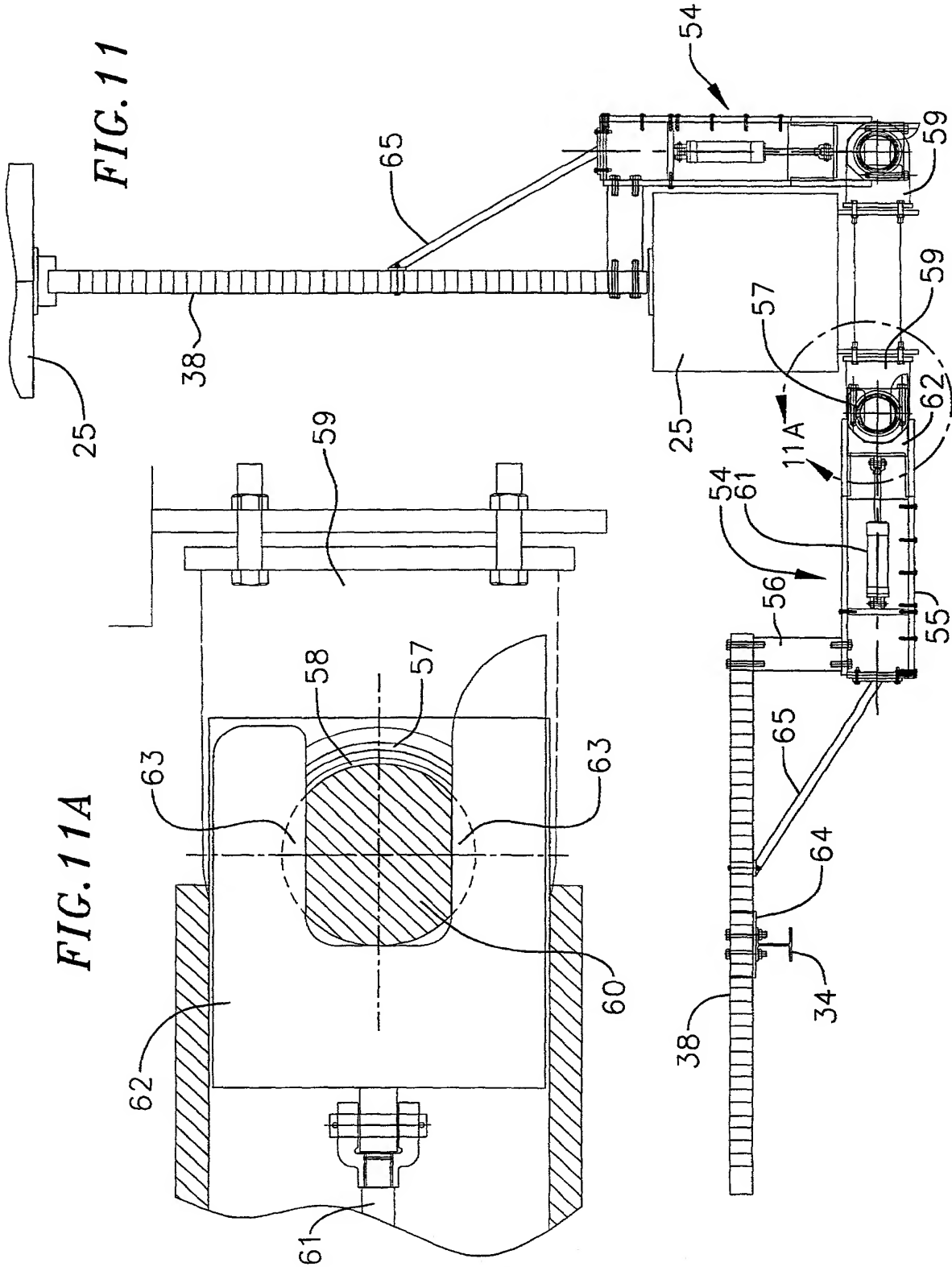


FIG. 12

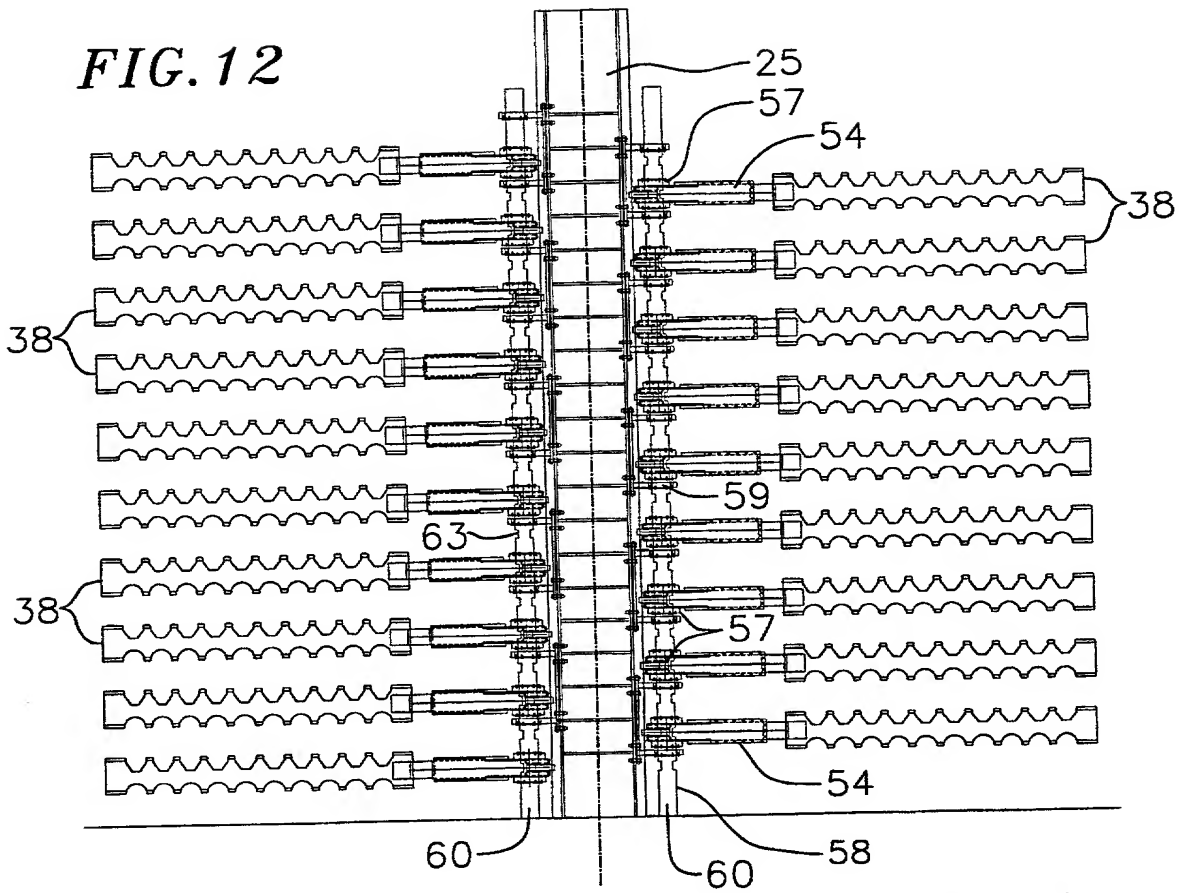


FIG. 13

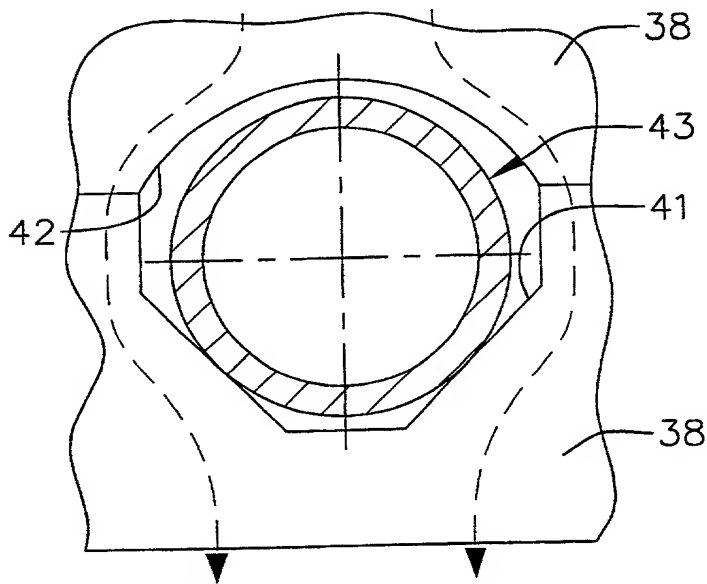
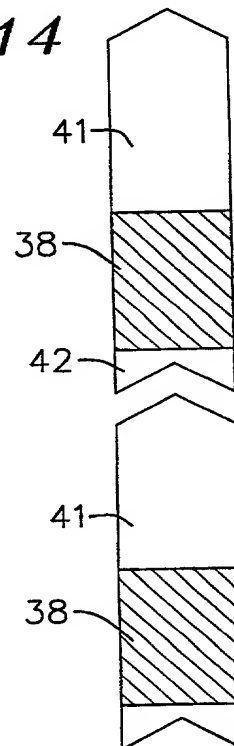
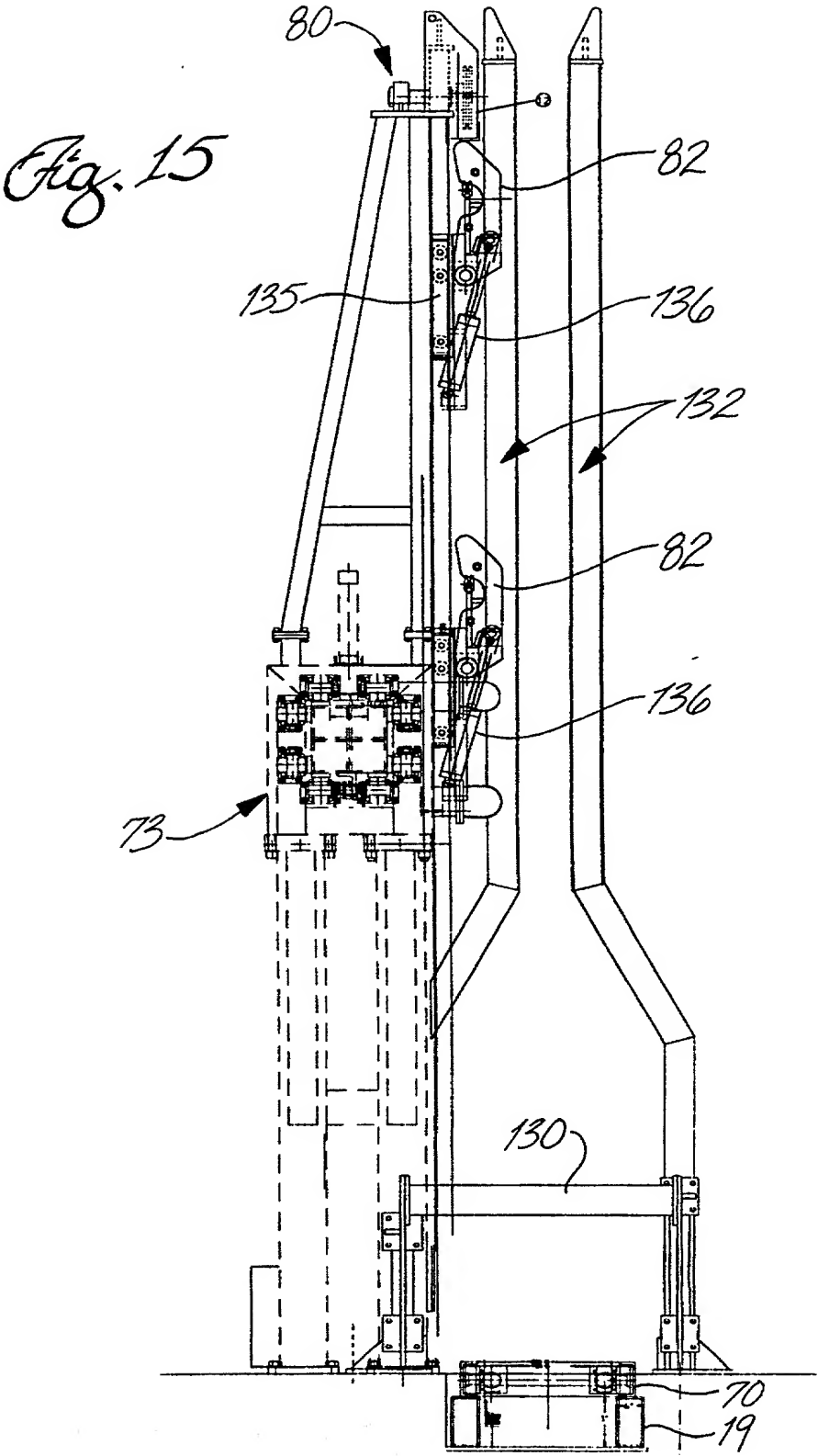


FIG. 14





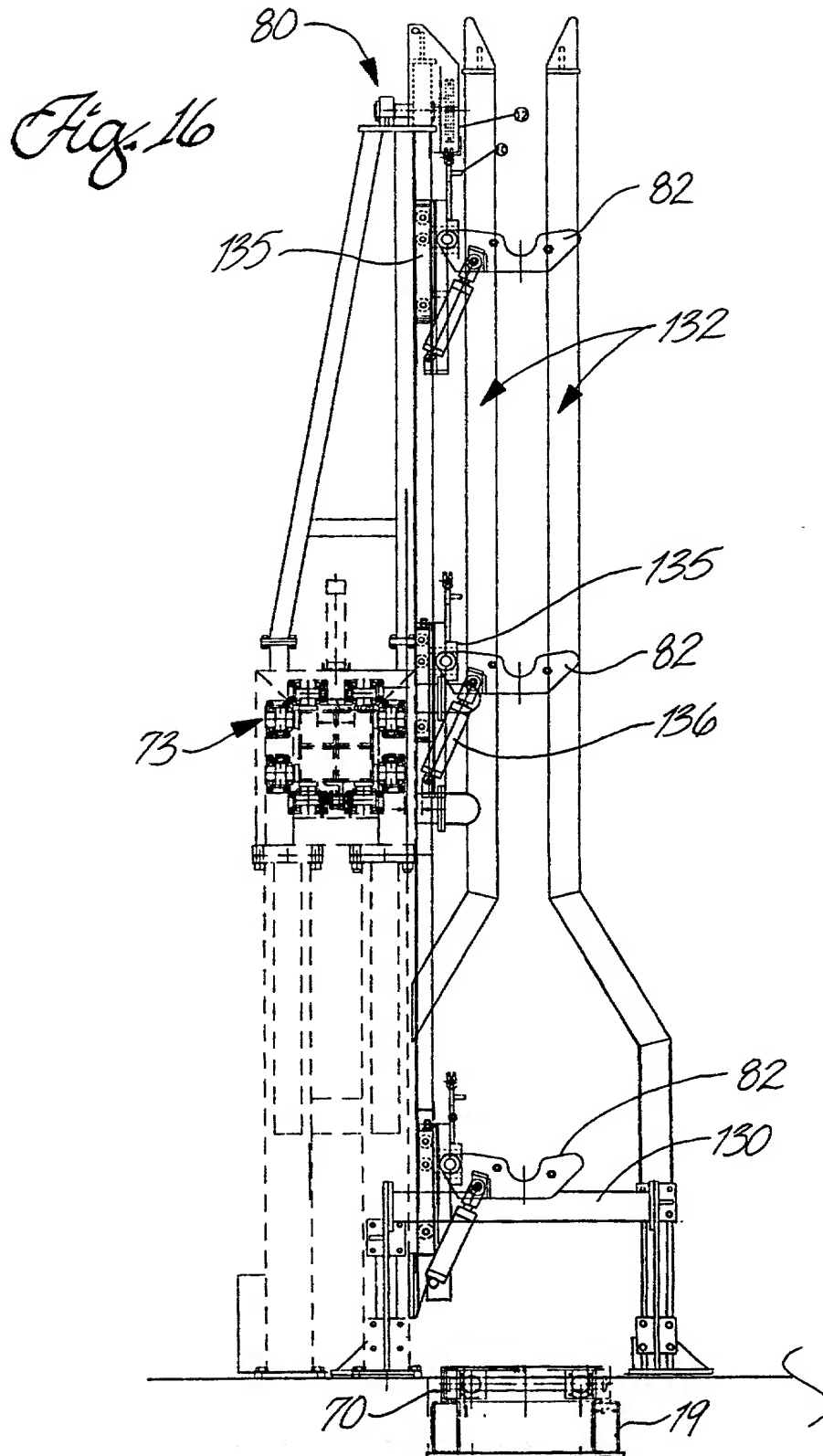


FIG. 17

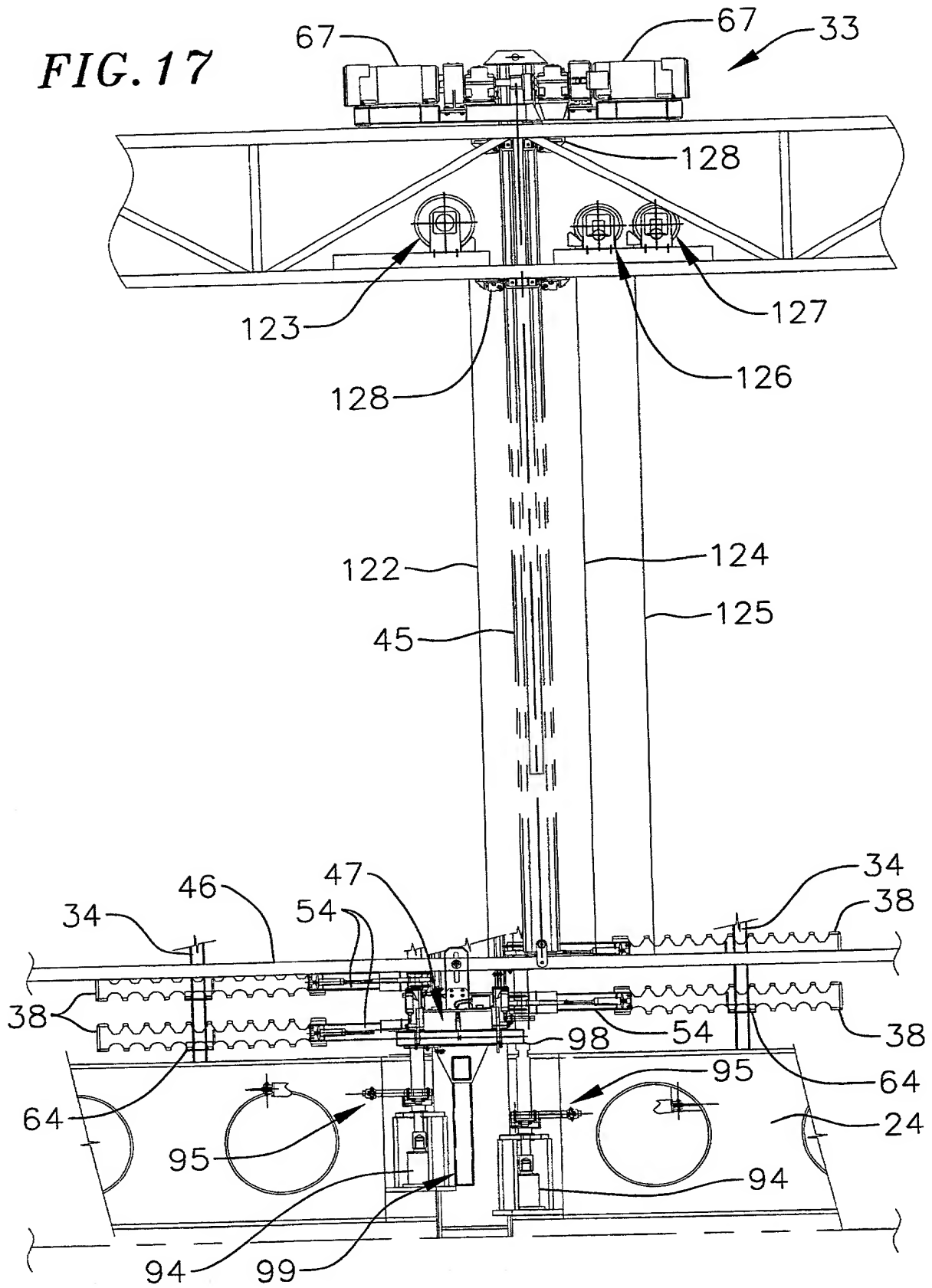


FIG. 18

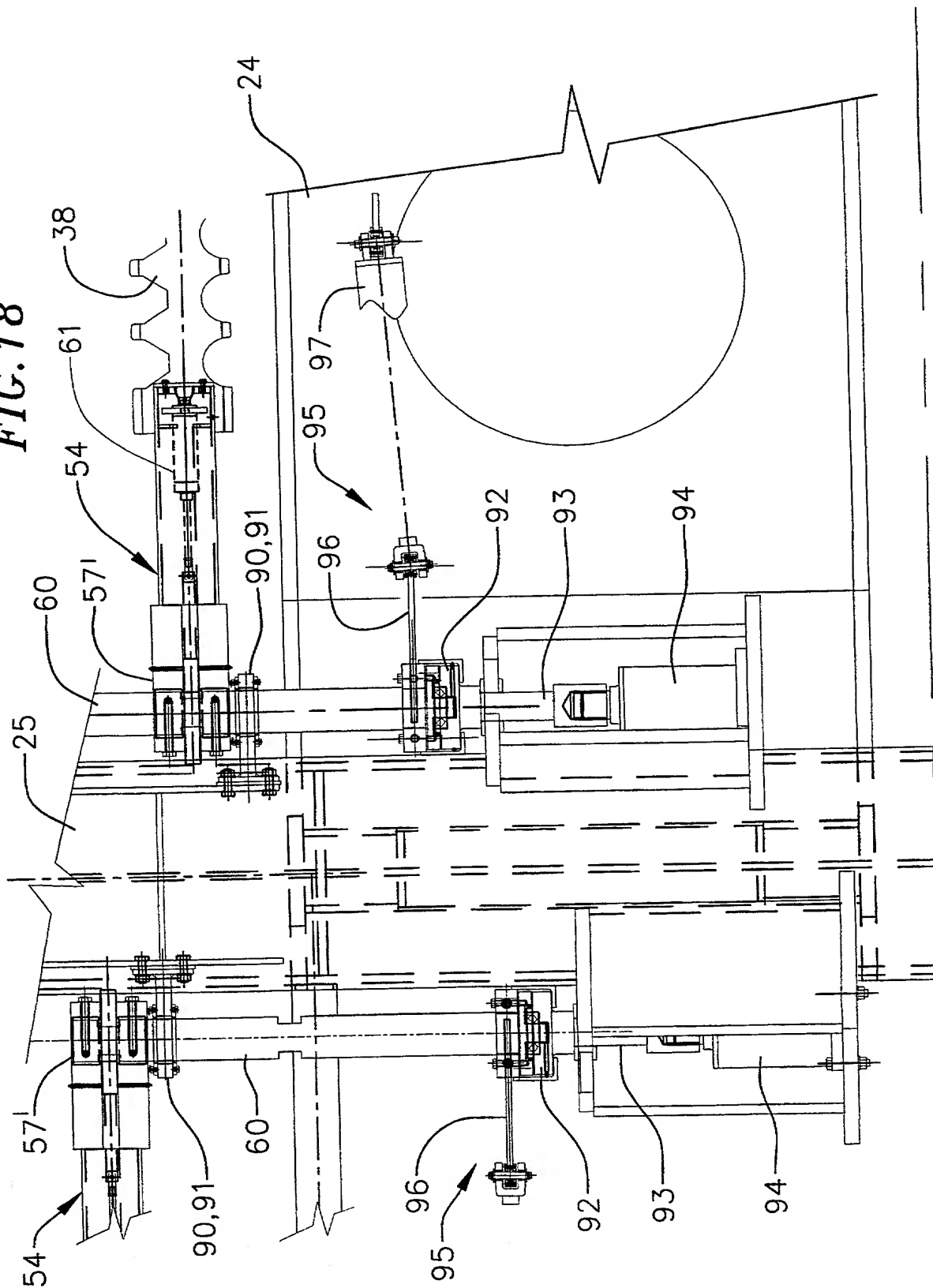


FIG. 19

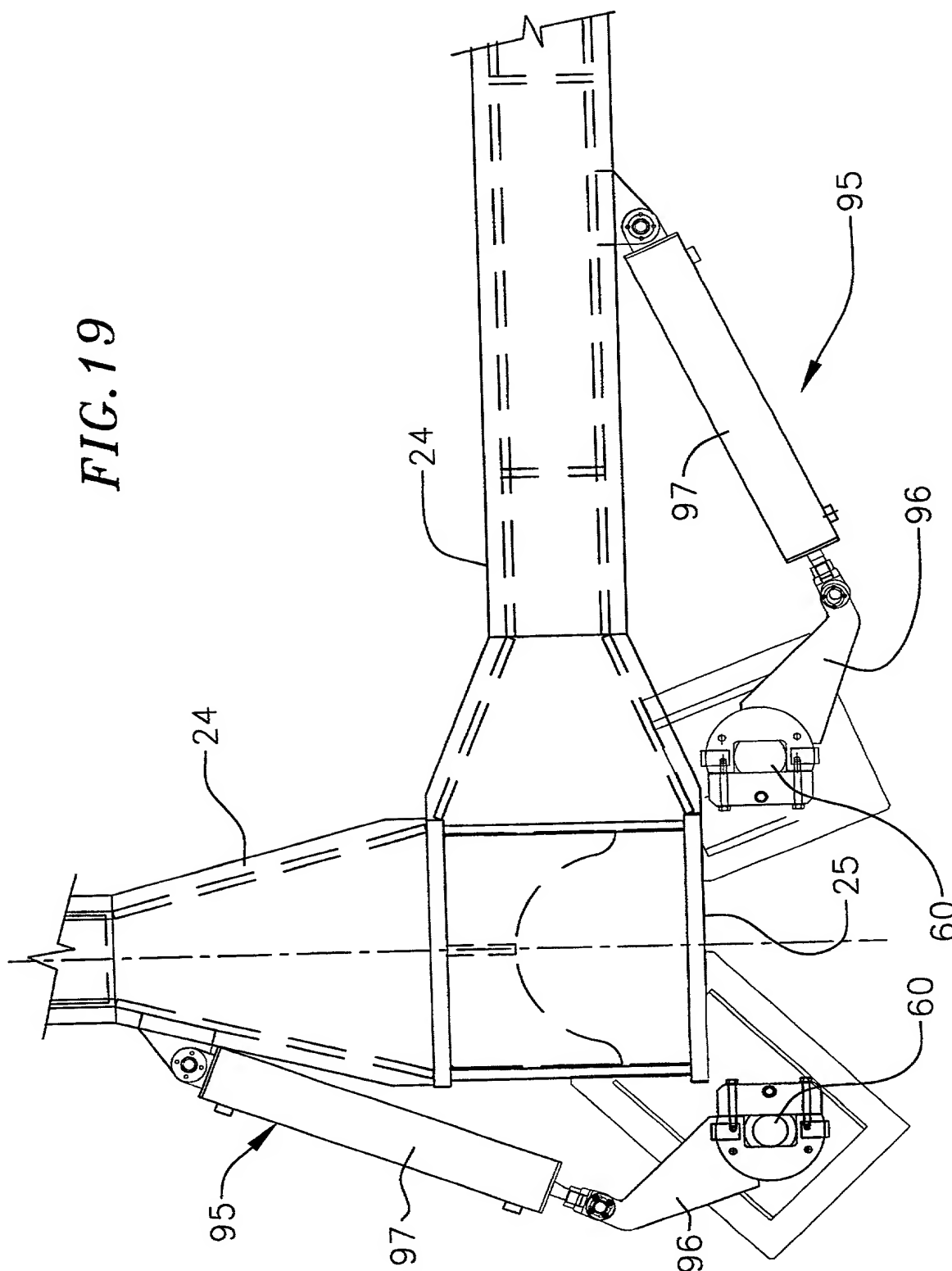


FIG. 20

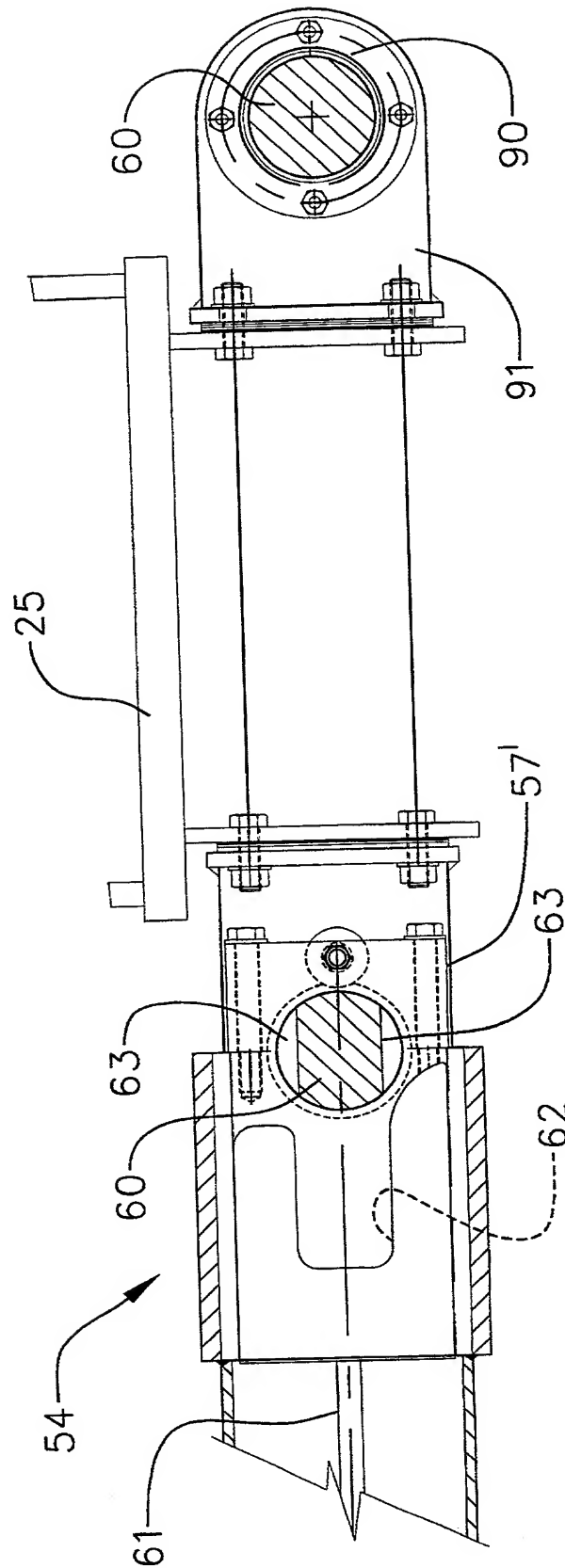


FIG. 21

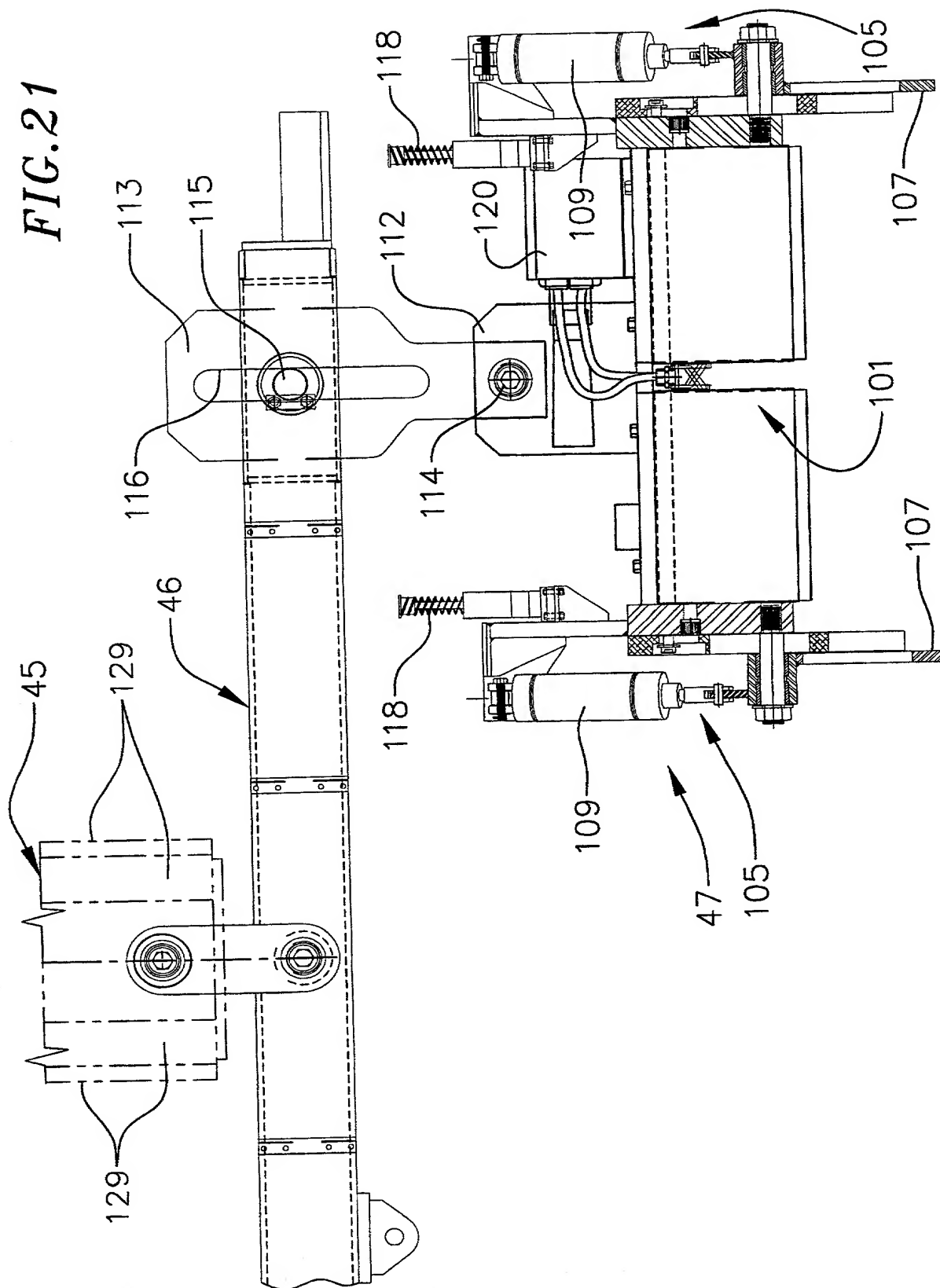


FIG.22

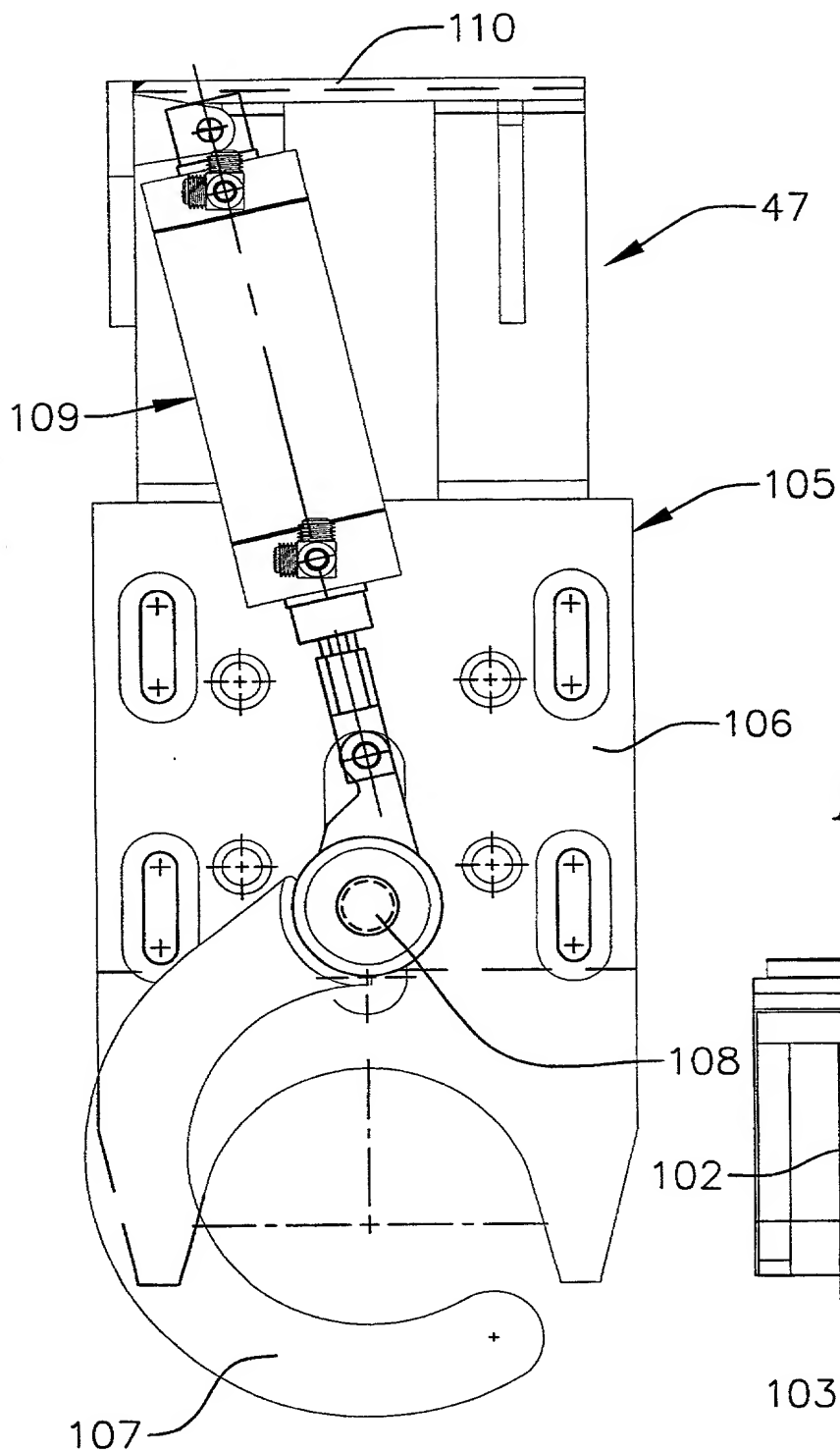
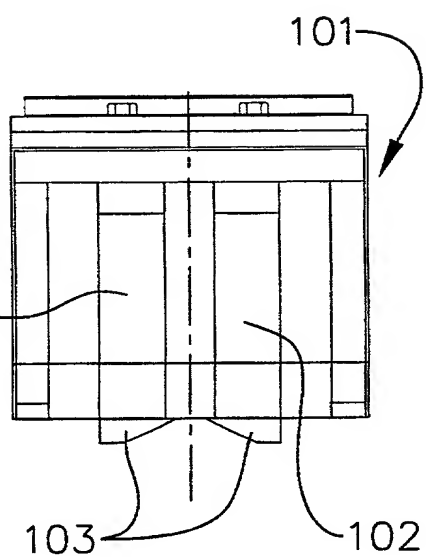


FIG.23



Rev 11/00

DECLARATION AND POWER OF ATTORNEY FOR PATENT APPLICATION

PATENT

Docket No. : 39373/HAC/G602

As a below named inventor, I hereby declare that:

My residence, mailing address and citizenship are as stated below next to my name.

I believe I am the original, first and sole inventor (if only one name is listed below) or an original, first and joint inventor (if plural names are listed below) of the subject matter which is claimed and for which a patent is sought on the invention entitled HORIZONTAL DRILL PIPE RACKER AND DELIVERY SYSTEM, the specification of which is attached hereto unless the following is checked:

___ was filed on ___ as United States Application Number or PCT International Application Number ___ and was amended on ___ (if applicable).

I hereby state that I have reviewed and understand the contents of the above-identified specification, including the claims, as amended by any amendment specifically referred to above.

I acknowledge the duty to disclose information which is material to patentability as defined in 37 CFR § 1.56, including for continuation-in-part applications, material information which became available between the filing date of the prior application and the national or PCT international filing date of the continuation-in-part application.

I hereby claim foreign priority benefits under 35 U.S.C. § 119(a)-(d) or § 365(b) of the foreign application(s) for patent or inventor's certificate, or § 365(a) of any PCT International application which designated at least one country other than the United States of America, listed below and have also identified below, any foreign application for patent or inventor's certificate, or any PCT International application having a filing date before that of the application on which priority is claimed.

Prior Foreign Application(s)

<u>Application Number</u>	<u>Country</u>	<u>Filing Date (day/month/year)</u>	<u>Priority Claimed</u>
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I hereby claim the benefit under 35 U.S.C. § 119(e) of any United States provisional application(s) listed below.

<u>Application Number</u>	<u>Filing Date</u>
60/156,662	September 29, 1999

I hereby claim the benefit under 35 U.S.C. § 120 of any United States application(s), or any PCT International application designating the United States, listed below and, insofar as the subject matter of each of the claims of this application is not disclosed in the prior United States or PCT International application in the manner provided by the first paragraph of 35 U.S.C. § 112.

<u>Application Number</u>	<u>Filing Date</u>	<u>Patented/Pending/Abandoned</u>
PCT/US00/27043	29 September 2000	Pending

POWER OF ATTORNEY: I hereby appoint the following attorneys and agents of the law firm CHRISTIE, PARKER & HALE, LLP to prosecute this application and any international application under the Patent Cooperation Treaty based on it and to transact all business in the U.S. Patent and Trademark Office connected

DECLARATION AND POWER OF ATTORNEY FOR PATENT APPLICATION

Docket No. 39373/HAC/G602

with either of them in accordance with instructions from the assignee of the entire interest in this application; or from the first or sole inventor named below in the event the application is not assigned; or from ___ in the event the power granted herein is for an application filed on behalf of a foreign attorney or agent.

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I declare that all statements made herein of my own knowledge are true and that all statements made on information and belief are believed to be true; and further that these statements were made with the knowledge that willful false statements and the like so made are punishable by fine or imprisonment, or both, under 18 U.S.C. 1001 and that such willful false statements may jeopardize the validity of the application or any patent issued thereon.

**DECLARATION AND POWER OF ATTORNEY
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